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Advances in fixed-bed reactor modeling using particle-resolved computational fluid dynamics (CFD)

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Abstract: In 2006, Dixon et al. published the comprehensive review article entitled “Packed tubular reactor modeling and catalyst design using computational fluid dynamics.” More than one decade later, many researchers have contributed to novel insights, as well as a deeper understanding of the topic. Likewise, complexity has grown and new issues have arisen, for example, by coupling microkinetics with computational fluid dynamics (CFD). In this review article, the latest advances are summarized in the field of modeling fixed-bed reactors with particle-resolved CFD, i.e. a geometric resolution of every pellet in the bed. The current challenges of the detailed modeling are described, i.e. packing generation, meshing, and solving with an emphasis on coupling microkinetics with CFD. Applications of this detailed approach are discussed, i.e. fluid dynamics and pressure drop, dispersion, heat and mass transfer, as well as heterogeneous catalytic systems. Finally, conclusions and future prospects are presented.

Keywords: CFD; fixed-bed reactor; fluid dynamics; heat transfer; surface chemistry.

Abbreviations

ADPF	axially dispersed plug flow
BCC	body-centered cubic
BET	Brunauer-Emmett-Teller
CAD	computer aided design
CFD	computational fluid dynamics
CPOX	catalytic partial oxidation

CT	computer tomography
DEM	discrete element method
DFT	density functional theory
DNS	direct numerical simulation
DRM	dry reforming of methane
FCC	face-centered cubic
FHS	front heat shield
LES	Large eddy simulation
LHHW	Langmuir-Hinshelwood-Hougon-Watson
PIV	particle-image velocimetry
RANS	Reynolds-averaged Navier-Stokes
RTD	residence-time distribution
SFR	stagnation-flow reactor
SRM	steam reforming of methane
SST	shear-stress transport
S2S	surface-to-surface
TST	transition-state theory

1 Introduction

Fixed-bed reactors are a widely used reactor type in the chemical and process industry. Among other applications, they play a key role for heterogeneous catalysis, e.g. steam and dry reforming of methane, the oxidative coupling of methane to ethylene, or the Sabatier process. Due to the strong endothermic (or exothermic) character of many kinds of surface reactions, heat needs to be effectively transferred into (or out of) the system. This leads to reactors with a small tube diameter. The particle size is restricted by several constraints like low pressure drop, high gas through-put, and high specific catalytic surface area. These restrictions lead to a reactor arrangement with a small tube-to-particle diameter ratio ($D/d_p = N$).

For fixed beds with a small N , conventional approaches like pseudo-homogeneous or heterogeneous models are not well suited, as they do not take into account local flow effects having a dramatic influence on fluid dynamics, as well as heat and mass transfer. For that reason, starting in the late 1990s, a considerable growing number of researchers developed methods to investigate the physical phenomena that take place in fixed-bed reactors by applying computational fluid dynamics (CFD) for three-dimensional

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(3D) particle-resolved simulations. This modeling approach takes into account the actual geometric structure in beds consisting of pellets. This means that the transport of momentum, heat, and species mass is resolved in the interstitial region of the pellets. Also, it is possible to resolve transport of heat and species mass in the interior of the pellets (intraparticle transport). As can be seen in Figure 1, the porous media model uses averaged values for the bed morphology. There is no clear distinction between the phases. Particle-resolved CFD simulations reconstruct the actual pellet shapes, which influences transport phenomena on a local basis. The corresponding porosity of the two approaches is illustrated in Figure 1C.

In 2006, Dixon et al. summarized the development of particle-resolved CFD simulations, which started in the mid-1990s. Many important aspects were covered, such as packing generation, meshing, and some applications of fluid dynamic problems including heat transfer, mass transfer, and chemical reactions. Due to the limited computational hardware at that time, the investigations were restricted to either periodic segments of a regular arranged packing or a small amount of particles (<50) forming a random fixed bed.

In the last 10 years, computer hardware has become much faster and more affordable. Furthermore, modern computer architecture makes it possible to connect multiple

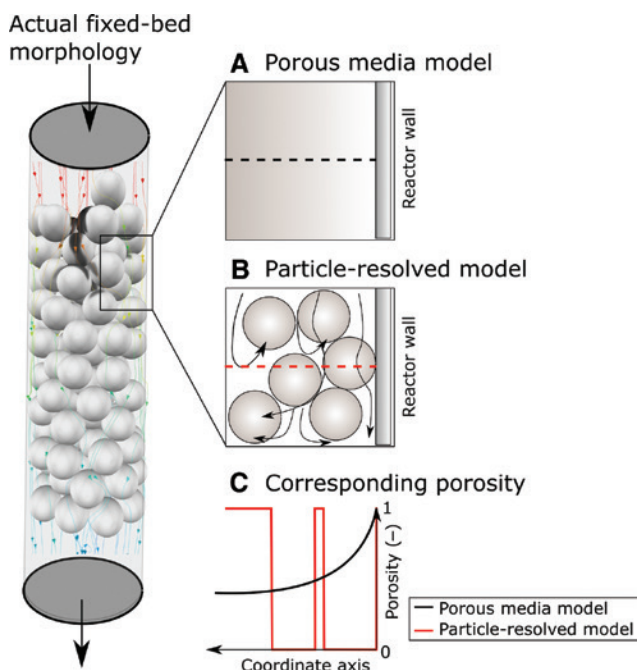


Figure 1: (A) Conventional porous media model and (B) particle-resolved CFD simulation of a fixed-bed reactor consisting of spherical pellets. (C) Corresponding porosity appears as dashed line through the bed.

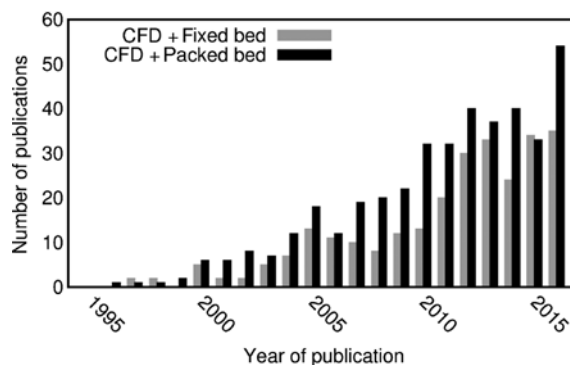


Figure 2: Number of publications per year searching article titles, abstracts, and article keywords with “CFD and fixed and bed” and “CFD and packed and bed” in the bibliographic database Scopus (2017) on 05/11/2017.

processor nodes to one high performance cluster (HPC). Together with the possibility of process parallelization, this leads to an intensified attraction using CFD in the field of chemical and process engineering in the last years, both in industry and academia. A growing part of this increased application covers numerical investigations of fixed-bed reactors. From the beginning, in the mid-1990s, almost 500 publications can be found on Scopus (2017) by searching for CFD and fixed bed or packed bed, see Figure 2, although not all of these publications use the particle-resolved model. On the one hand, this is a true indicator that there are still open questions that need to be answered. On the other hand, it shows that CFD has been developed to a useful tool that helps to gain in-depth insights of complex reactor devices.

This work summarizes the advances that have been made within the last decade in the field of fixed-bed reactor modeling. Earlier development was reviewed comprehensively by Dixon et al. (2006). We show and discuss recent results, new and improved modeling approaches, and limitations that still exist. Furthermore, some current best practices are derived. The next section will discuss challenges during a typical workflow that needs to be tackled for a successful CFD simulation. The third section is addressed to the discussion of recent applications of particle-resolved CFD simulations. Finally, conclusions are drawn and future prospects are presented.

2 General workflow and challenges

The general workflow for particle-resolved CFD simulations of fixed-bed reactors is presented in Figure 3 including the corresponding chapters of this review. The first step is the generation of a representative geometry, which can be based on a scanned original sample, a regular

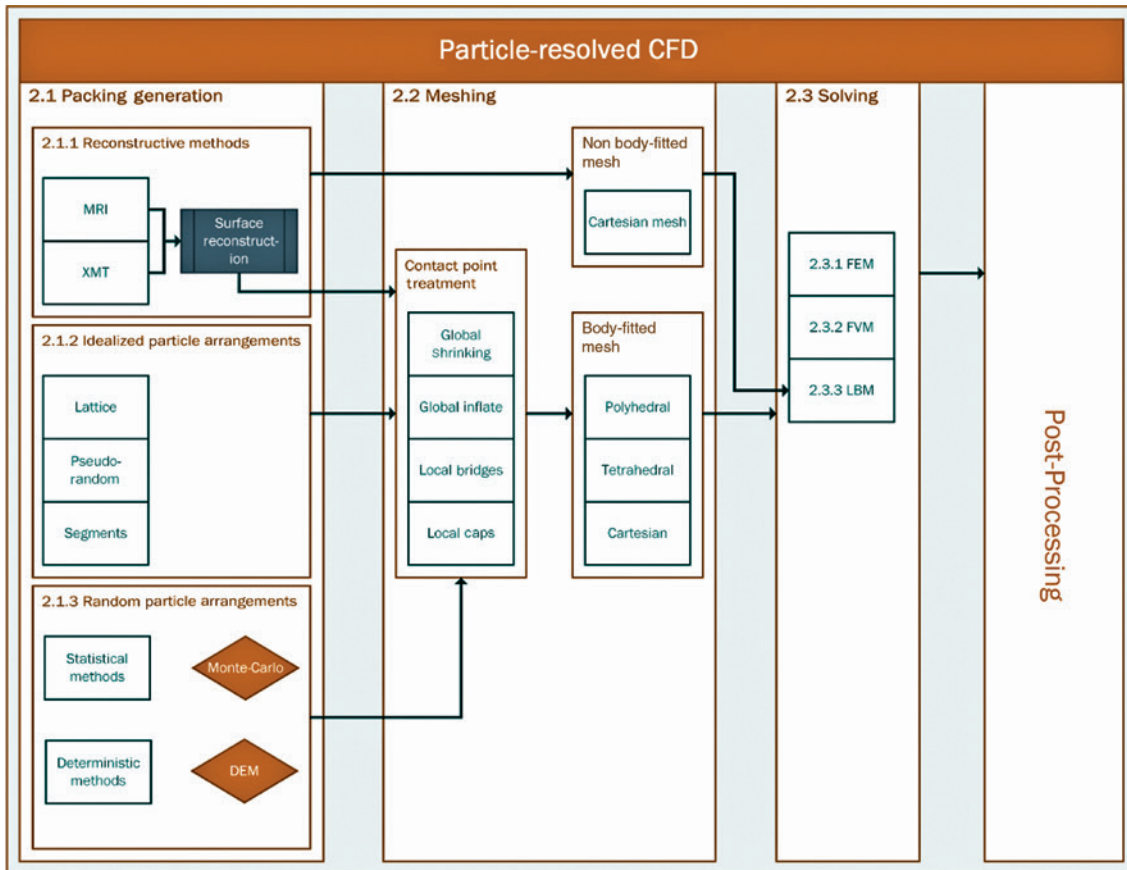


Figure 3: General workflow of particle-resolved CFD simulations with corresponding sections of this review article.

arrangement, or a synthetically generated bed structure. In the second step, a discretization procedure of the calculation domain is carried out, as the Navier-Stokes equations have to be solved iteratively. Depending on the numerical method, a mesh or a number of grid points is generated. If chemical reactions are also of interest, coupling between flow field, species concentrations, temperature distribution, and the chemical kinetics is needed. Data analysis and extraction, as well as visualization of the results, are the final step in the workflow. All those steps are accompanied by specific challenges, which need to get mastered. Many of them are discussed in the following sections.

2.1 Packing generation

The first aspect to consider for a particle-resolved CFD simulation is the geometrical representation of the fixed bed. In a consecutive step, this information is transferred into CAD data. The geometrical representation can be very close to a specific packing, which can be achieved with

all kinds of scanning techniques and numerical methods, or very general, which is the case for unit-cell models. All of these geometrical representatives have their advantages and disadvantages, which will be discussed in the following.

2.1.1 Reconstructive methods

In an experiment, tubes or other kinds of containers are filled with particles leading to a random bed. With 3D reconstruction techniques, the actual shape, position, and orientation of each particle are gathered. Tomographical methods like magnetic resonance imaging (MRI) or X-ray microtomography (XMT) have been applied by several authors, e.g. Wang et al. (2001), Baker (2011), and Yang et al. (2013). The output of tomography methods is voxel data that need to be transferred either to a surface description or directly to a volume representation of the numerical domain. The latter is less complicated to implement, as a voxel is simply treated as a volume cell. However, a non-body-fitted mesh is created, which is not state-of-the-art

of CFD anymore. For the creation of a smooth surface description from the voxel data, sophisticated reconstruction methods are needed, for a more detailed description, see the study by Yang et al. (2013).

The benefit of tomography methods is that almost any particle shape can be reconstructed and that the geometrical description of the bed morphology is almost identical to the original. The disadvantage is the high time consumption of the scanning and the reconstruction.

2.1.2 Idealized particle arrangements

The simplest kind of a bed structure is the regular arrangement. An explicit mathematical description of the position of each individual particle can be derived. Early research on particle-resolved CFD simulations was carried out in such regular beds, for a review, see the study by Dixon et al. (2006). More recently, several authors used regularly arranged beds to study different physical aspects in detail. Lee et al. (2007), Shams et al. (2013a,b, 2014, 2015), and Ferng and Lin (2013) investigated different levels of detail for turbulence and corresponding heat transfer, i.e. direct numerical simulation (DNS), large-eddy simulation (LES), and several different Reynolds-averaged Navier-Stokes (RANS) models. Unit-cell models are one of the smallest investigated sections of fixed beds. Typically, simple cubic (SC), face-centered cubic (FCC), and body-centered cubic (BCC) unit cells are compared. Bu et al. (2014) studied the convective heat transfer using different arrangements of spherical particles in a unit cell, i.e. SC, FCC, and BCC, and compared different particle-particle contact-point modifications. Dixon et al. (2013a) showed that a pseudo-random packing can be achieved by a spiral arrangement of six layers each consisting of 12 spheres. The impact of non-spherical particles on the heat transfer has been studied by Yang et al. (2010) for FCC-ordered ellipsoids in a rectangular channel and by Dixon et al. (2008) for cylindrical shape types with inner voids. Dixon et al. (2007, 2012a), Behnam et al. (2012), and Cheng et al. (2010) used stacked particles or representative segments to study catalytic steam reforming of methane (SRM) on spherical and non-spherical particles.

The advantage of regularly arranged beds is the fast and simple generation of a geometric representation. Unit-cell models are ideal representatives of bed sections. Conclusions gained from such investigated structures can be extrapolated to random beds. Especially for spherical particles, regular arrangements are often present in randomly filled containers. However, for non-spherical particles, it is more complicated to build up generalized

structures. Consequently, idealized arrangements can be used for benchmark investigations serving as a validation database. The decreasing computational effort allows a reduction of modeling assumptions, as was applied in a series of investigations by Shams et al. (2013a,b, 2015).

2.1.3 Random particle arrangements

The previously presented methods for packing generation are either not suited to create beds with randomly arranged particles or too expensive and time-consuming to find a wider application, also in the industry. Therefore, methods for a synthetic generation of representative random beds have been in the focus since the beginning of particle-resolved CFD. Dixon et al. (2006) classified packing strategies into sequential deposition algorithms and collective rearrangement methods. The former include drop-and-roll techniques and the one-by-one placement of particles based on pre-defined seed pellets or clusters. The collective rearrangement is basically a statistical Monte Carlo method. Particles are initialized randomly in the domain and afterwards statistically moved to either reduce overlaps or minimize void fraction. While the sequential deposition algorithms almost vanished in the last years, statistical methods still play an important role. Nowadays, all packing algorithms are either statistical or deterministic approaches.

The statistical methods are mostly Monte-Carlo-based methods where, in a first step, a number of particles are randomly distributed in the numerical domain. Different methods have been developed to generate the final bed morphology out of this point cloud. Atmakidis and Kenig (2009, 2012) used an approach where after each injection step, only the particle is kept with the lowest position not intersecting with another particle. A comparison of the radial void fraction profile and the overall porosity with correlations by de Klerk (2003) showed that the numerical algorithms tend to create less dense beds. Furthermore, the radial void fraction distribution showed a much more damped oscillating behavior. This has also been reported by Auwerda et al. (2010) when simple Monte Carlo approaches were used. The authors compared a Monte Carlo rejection method developed by Kloosterman and Ougouag (2007) against the expanding system code established by Mrafsko (1980). The rejection method is a bottom-up approach that deletes all particles that overlap after the initial injection step. The expanding system code inflates the particles step-by-step until the final particle size is reached. Particles are moved apart, which get

in contact during this procedure. It is shown that the rejection method is not able to deliver accurate results regarding bed porosity and radial void fraction distribution, whereas the expanding system algorithm leads to satisfactory results.

A drawback of Monte Carlo methods that does not take particle collisions into account is that the algorithms lead to nonphysical particle arrangements for non-spherical particles or if reactor internals are present, cf. Caulkin et al. (2009a). Consequently, Caulkin et al. (2009a, 2012) extended the method and compared the Monte Carlo code DigiPac against the hybrid code DigiCGP (Collision Guided Packing) regarding porosity profiles of fixed beds made of spherical and non-spherical particles. DigiCGP takes particle collisions into account. Colliding voxels are detected and assigned to a nominal impact force of one pointing at the direction along a connecting line between the center of gravity of each particle and the collision point. For the torque calculation, the impact-force vector is assumed to be normal to the contact face between the colliding voxels. The torque vector itself is used to determine the rotation axis. The rotation is modeled by applying a random twist angle. It is shown that the hybrid approach is able to predict global bed porosity and void fraction profiles within a deviation of 4% for cylindrical-like particles in reactors including internals. The classical method, however, fails. While the void fraction is satisfactorily reproduced by this algorithm, Caulkin et al. (2008, 2009b) showed that the local particle orientation was not in agreement.

The deterministic discrete element method (DEM) has shown more promising results. DEM is an engineering approach to simulate many moving discrete particles that interact with each other and the surrounding flow. It is an extension of the Lagrangian modeling approach, which was established by Cundall and Strack (1979) and is nowadays implemented in numerous commercial software packages (e.g. STAR-CCM+, ROCKY DEM, EDEM, or PFC) and open-source codes like LIGGGHTS or YADE-DEM. The basic idea of DEM is to include inter-particle contact forces into the equations of motion. Two DEM frameworks can be distinguished: the hard-sphere and the soft-sphere frameworks. In the hard-sphere model, the particles are ideally elastic, and the particle collisions are instantaneous. The benefit of this assumption is that a temporal resolution of the collision mechanism can be avoided and numerical costs are reduced. However, it is only valid if the system is not dominated by multi-particle contacts, i.e. dilute particle regimes. Recently, Boccardo et al. (2015) used a hard-sphere approach by using the Bullet Physics library in Blender [see Blender-Foundation (2015)] to

generate beds with spherical, cylindrical, and trilobe particles achieving satisfactory results.

The more general model is the so-called soft-sphere model where the particles are allowed to overlap and the contact forces are proportional to the overlap, particle material, and geometrical properties. The interaction between particles and between particles and walls is determined by the momentum balance equation for a material particle:

$$m_p \frac{d\mathbf{v}_p}{dt} = \mathbf{F}_s + \mathbf{F}_b, \quad (1)$$

where \mathbf{F}_s is the forces acting on the particle's surface, i.e. drag force, and pressure gradient force, and \mathbf{F}_b is the body forces:

$$\mathbf{F}_b = \mathbf{F}_g + \mathbf{F}_c \quad (2)$$

with \mathbf{F}_g as the gravity force and \mathbf{F}_c as the contact forces, which are defined as follows:

$$\mathbf{F}_c = \sum_{\text{neighbor particles}} \mathbf{F}_{\text{contact}} + \sum_{\text{neighbor walls}} \mathbf{F}_{\text{contact}}. \quad (3)$$

To model the forces, several approaches exist, e.g. the linear spring model, the non-linear spring-dashpot model by Hertz-Mindlin, or the Walton-Braun hysteretic linear spring model. An extensive description of different models is given by Zhu et al. (2007) and Di Renzo and Di Maio (2000).

Most DEM codes use the above-discussed algebraic contact-detection algorithms based on mass points that work in a Lagrangian framework. Despite of that, two different methods exist that Caulkin et al. (2015) call voxel-based DEM and surface mesh DEM. For the voxel method, each particle is discretized by a number of voxels, which are allowed to overlap. When they collide, a restitution force proportional to the overlap volume is calculated based on a linear spring-dashpot model. Xu et al. (2006) and Caulkin et al. (2015) used voxel-based DEM in their work to create beds of cylindrical particles and achieved good results regarding void fraction and particle orientation compared to NMR/XMT measurements. For the surface mesh based particle collision model, the particles are represented by vertices, edges, and faces. Based on intersections of the surface representation of the particles, the restitution forces during a collision are calculated. Marek (2013) and Niegodajew and Marek (2016) used the latter approach to create packings of cylindrical particles and Raschig rings. Also the particle interaction model in Blender used by Boccardo et al. (2015) is based on this method.

The trend to use DEM as a tool for packed-bed generation is a quite new one. Dixon et al. (2006) reported in their review article only about the work presented by Theuerkauf et al. (2006), who used DEM to create random packings of spherical particles. Since then, the use of DEM became predominant to create beds of spherical and non-spherical particles. Earlier works of Ookawara et al. (2007) and Kuroki et al. (2007) used DEM to create beds of spherical particles and could show that the global void fraction is in good agreement with experimental values. Furthermore, they found that the friction factor can be used to tune the void fraction to a certain value. This can be useful if a vibration-induced artificially compacted bed morphology is wanted.

While in the early years, the use of DEM was limited to spherical particles, for which the original model was formulated, nowadays, it can also be used for non-spherical shapes. Figure 4 shows different approaches that can be applied to model non-spherical particles. One of the earliest developments was the approximation by a so-called composite particle. The desired shape of the non-spherical body is approximated by a user-defined amount of spheres, which retain their position with each other. That composite particle can be created either manually or by the use of Monte-Carlo-based automation methods. For

simple shapes, it is beneficial to create the composite particle manually, as it is a more efficient method. If the shape gets more complex, an automated procedure can be applied. It has to be mentioned that the more DEM spheres are used to approximate the desired shape, the more computational time is required. Furthermore, the edges of the original particle shape are not represented, as the shape is approximated with spheres. This is a disadvantage of the composite particle approach. Kodam et al. (2010a,b) developed and validated a sophisticated contact detection algorithm for genuine cylindrical particles. For different contact scenarios, they deduced equations to calculate the overlap, location, and normal vector of the contact point between two cylindrical particles or between a particle and a planar wall. Feng et al. (2017) generalized this approach and developed a framework where a full exploitation of the axisymmetrical property of the cylinders is used to detect the contacts. Their method also works for the interaction of cylinders with spheres or half-spheres.

The first packing of cylindrical particles was presented by Bai et al. (2009). The authors used the composite-particle approach and generated a bed consisting of 82 particles investigating fluid dynamics. If composite particles are used, a crucial point for the accuracy is the number of DEM particles approximating the original shape.

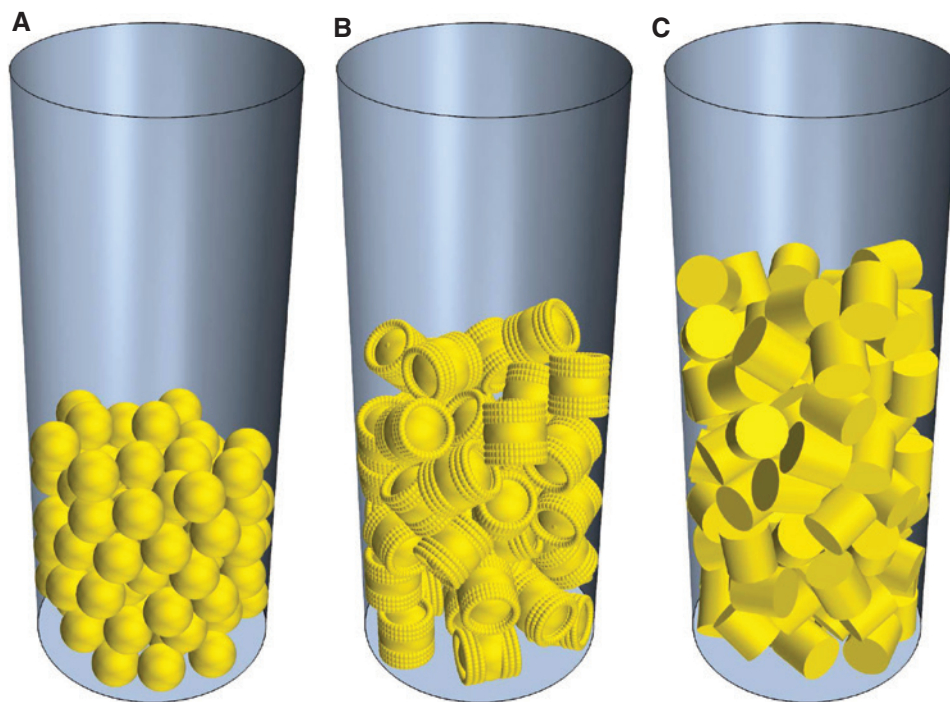


Figure 4: Different kinds of DEM particles forming a packed bed. (A) Spherical. (B) Composite and (C) Cylindrical.

Caulkin et al. (2015) compared different particle representations that differed in their edge roundness, surface roughness, and restitutional behavior with experimental results. It was shown that an adequate approximation of sharp edges is needed to get satisfying results. Seventy-nine spheres per shape were needed for an acceptable characterization of the particle behavior.

In the last years, DEM has been extensively applied by several authors to create fixed beds with particles of different shapes. Table 1 summarizes the most important work on numerical methods in the field of fixed-bed reactors with random particle arrangement. Special attention is paid to particle shape and packing method and whether the generated bed morphology was validated against experimental results or against correlations.

2.2 Meshing

The discussed packing-generation methods provide either information on the particle position and orientation or a voxel representation of the bed geometry is given. In the first case, an overall CAD model of the fixed bed can be generated by placing one by one a CAD description of the individual particle based on the position and orientation data. If voxel data are used, either a surface-reconstruction algorithm needs to be applied for creating CAD data or the voxel data can directly be utilized as a non-body-fitted hexahedral volume mesh [see Yang et al. (2013)].

Depending on the kind of numerical solver, the geometrical representation needs to get spatially discretized by a mesh with cells of a specific type. Different mesh types are depicted in Figure 5. Fundamentally, two categories of meshes can be distinguished, i.e. structured and unstructured meshes. Figure 5A shows an example of a curvilinear structured grid where every node is explicitly defined according to the specified algorithm. That mesh type is numerically very efficient but obviously not applicable for complex geometries like fixed-bed reactors. For that kind of applications, unstructured meshes like the ones depicted in Figure 5B–F are much more advantageous. For different numerical methods, other mesh types are applicable or preferred.

For simulations based on the Lattice-Boltzmann method (LBM), typically Cartesian meshes are applied. At curved boundaries, a cut-cell approach according to Figure 5D can be used to retain the geometrical features. This is an efficient meshing strategy especially for complex geometries. For LBM, there is no need to calculate cell face fluxes. This method also works stably if the trimmed cells collapse at the particle-particle and particle-wall contact

points. For the finite-element method (FEM), most often, tetrahedral or hexahedral meshes similar to Figure 5B and C are used. For finite-volume method (FVM) codes, all mesh types depicted are at least applicable, besides the Cartesian cut-cell approach. Although many authors, as shown in Table 1, still use tetrahedral meshes, Peric (2004) showed that polyhedral cells tend to have less numerical dissipation. In addition, they converge faster compared with tetrahedral cells and are well suited for complex geometries involving tortuous flow. It is the best practice to use layers of prismatic cells, as can be seen in Figure 5F, at all fluid wall boundaries including the particles to resolve the boundary layers.

In reality, randomly packed particles inside a reactor tube touch their neighboring particles or the tube wall. Between spherical particles, contact points occur. Contrarily, non-spherical particles stay in contact with other particles or the wall with contact points, lines, or areas, see Figure 6. Whatever cell type is chosen, the meshing close to particle-particle and particle-wall contacts is a true challenge. If no modification is carried out, the low cell quality at the contacts will lead to serious convergence issues during the simulation. Figure 6 summarizes different modes of contacts in packed beds and common modification strategies.

The modifying approaches can be classified into global and local geometrical modifications. The global shrinking method was one of the first approaches that have been developed and reported for beds of spheres (Esterl et al. 1998). All particles are shrunk by a certain amount leading to a small gap, which can be filled with volume cells of reasonable quality. This technique, also known as the global *gaps* method (Dixon et al. 2013b), has been utilized by several authors within the last years, see Table 1. Guardo et al. (2004) developed an alternative approach and inflated the spherical particles by a certain degree. As a consequence, the angle between pointy faces at the contact point is increased. The vicinity, where faces intersect, can be filled with cells of good quality. This method was later referred to as the global *overlaps* method (Dixon et al. 2013b). The global increasing or decreasing of the particle size is a critical step, as it modifies the geometric representation significantly. This can be easily understood by the following estimate: taking the pressure drop correlation by Ergun (1952) in the fully turbulent flow regime for a porosity of $\varepsilon = 0.4$, changing the void fraction by 2.5% results in a deviation regarding the pressure drop of 10%. As the porosity is proportional to the particle diameter by $\varepsilon \propto d_p^3$, this means that changing the particle size by 1.35% will change the porosity by 2.5% and leads to a deviation of the pressure drop of

Table 1: Publications on randomly packed beds of the last 10 years.

Publication	Particle shape	Packing method	Mesh type	Contact modification	Solver	Morphology validation
DEM-based work						
Augier et al. (2010)	Sphere	Soft-sphere DEM	Tetrahedral	Global shrinking	FVM	Yes
Bai et al. (2009)	Sphere, cylinder	Soft-sphere DEM	Tetrahedral	Global shrinking	FVM	No
Behnam et al. (2013)	Sphere	Soft-sphere DEM	Tetrahedral	Global shrinking	FVM	Yes
Boccardo et al. (2015)	Sphere, cylinder, trilobes	Surface mesh based	Hexahedral	Local bridges	FVM	Yes
Caulkin et al. (2009b)	Cylinder, Raschig ring, Pall ring	Pixel-based DEM	–	–	–	Yes
Caulkin et al. (2015)	Cylinder	Soft-sphere DEM	–	–	–	Yes
Dixon et al. (2012b)	Sphere	Soft-sphere DEM	Tetrahedral	Global shrinking, local bridges	FVM	Yes
Eppinger et al. (2011)	Sphere	Soft-sphere DEM	Polyhedral	Local caps	FVM	Yes
Eppinger et al. (2014b)	Cylinder, Raschig ring, four-hole cylinder	Soft-sphere DEM	Polyhedral	Local caps	FVM	Yes
Eppinger et al. (2016)	Sphere	Soft-sphere DEM	Polyhedral	Local caps	FVM	No
Kuroki et al. (2007)	Sphere	Soft-sphere DEM	Tetrahedral	Local bridges	FVM	No
Kuroki et al. (2009)	Sphere	Soft-sphere DEM	Tetrahedral	Local bridges	FVM	No
Ookawara et al. (2007)	Sphere	Soft-sphere DEM	Tetrahedral	Local bridges	FVM	Yes
Rebughini et al. (2017)	Sphere	Soft-sphere DEM	Tetrahedral	Local bridges	FVM	No
Theuerkauf et al. (2006)	Sphere	Soft-sphere DEM	Tetrahedral	Local bridges	FVM	No
Toutou et al. (2014)	Sphere	Soft-sphere DEM	–	–	–	Yes
Wehinger et al. (2015a)	Sphere	Soft-sphere DEM	Tetrahedral	Not specified	FVM	No
Wehinger et al. (2015b)	Sphere	Soft-sphere DEM	Polyhedral	Local caps	FVM	Yes
Wehinger et al. (2015b)	Sphere, cylinder, Raschig ring	Soft-sphere DEM	Polyhedral	Local caps	FVM	No
Wehinger et al. (2016b)	Sphere	Soft-sphere DEM	Polyhedral	Local caps	FVM	Yes
Wehinger et al. (2017a)	Cylinder	Soft-sphere DEM	Polyhedral	Local caps	FVM	No
Xu et al. (2006)	Cylinder	Pixel-based DEM	–	–	–	Yes
Monte-Carlo-based work						
Auwerda et al. (2010)	Sphere	Monte-Carlo	–	–	–	Yes
Atmakidis and Kenig (2009)	Sphere	Monte-Carlo	Tetrahedral	Global shrinking	FVM	Yes
Atmakidis and Kenig (2012)	Sphere	Monte-Carlo	Tetrahedral	Global shrinking	FVM	No
Caulkin et al. (2008)	Sphere, cylinder, Raschig rings	Monte-Carlo, Hybrid	Cartesian	Not needed	LBM	Yes
Caulkin et al. (2009a)	Cylinder, Raschig ring, four-hole cylinder	Monte-Carlo, Hybrid	–	–	–	Yes
Caulkin et al. (2009b)	Cylinder, Raschig ring, Pall ring	Hybrid	–	–	–	Yes
Caulkin et al. (2012)	Cylinder, Raschig ring, trilobe	Hybrid	Cartesian	Not needed	LBM	Yes
Scanned geometries						
Baker (2011)	Cylinder	MRI	Hexa- and tetrahedrals	Local bridges	FVM	No
Motlagh and Hashemabadi (2008)	Cylinder	Photography	Tetrahedral	No modification	FEM	No
Yang et al. (2013)	Sphere	MRI	Hexahedral, polyhedral	Local caps	FVM	No

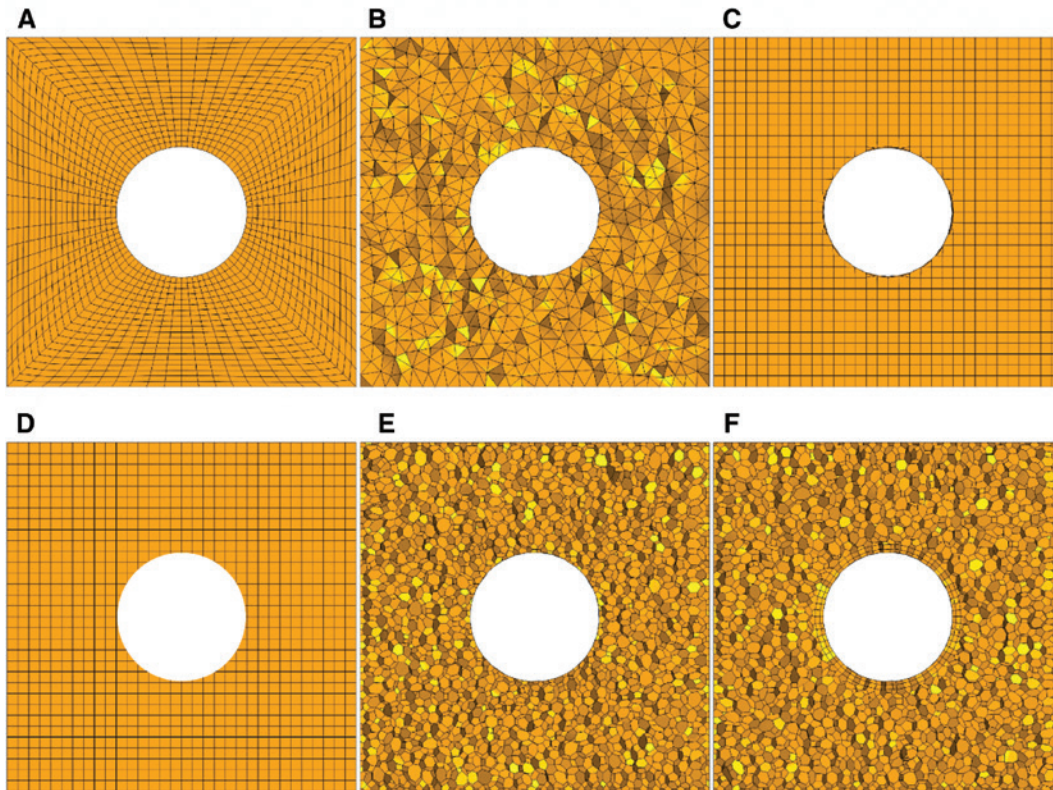


Figure 5: Different mesh types.

(A) Curvilinear structured. (B) Tetrahedral. (C) Mixed hexahedral. (D) Cartesian cut-cell. (E) Polyhedral and (F) Polyhedra with prism layers.

10%. Several authors encountered that issue. Augier et al. (2010) and Atmakidis and Kenig (2009) reported deviations of more than 15% regarding the calculated pressure drop in comparison with correlations. Bai et al. (2009) avoided the problem by introducing a porosity correction factor. That might be practicable, if one is just interested in the pressure drop of the system. However, it will lead to significant errors for properties that rely on a correct predicted flow field. This indicates that the geometrical modifications to avoid the contact problem should be reduced to a minimum.

Therefore, Ookawara et al. (2007) and Eppinger et al. (2011) proposed local acting geometry modifications. The former developed a local strategy by placing small cylinders between the center of touching spheres, i.e. the local *bridges* method. The particles and bridging cylinders were united afterwards and subtracted from the container to extract the fluid volume. On the contrary, Eppinger et al. (2011) introduced small voids in the vicinity of the contact points in a bed of spheres by a sophisticated surface-meshing technique, i.e. the local *caps* method. During the surface re-meshing process, the algorithm detects, if

faces are in proximity to each other. If a certain threshold is exceeded, the meshing algorithm projects the vertices along their connecting line to reach the specified threshold. The resulting space is filled with a defined number of volume cells of good quality. It should be avoided to fill the gaps with an unnecessarily large number of cell layers. Instead, only two layers should be used. As each layer is adjacent to a no-slip boundary condition, this strategy prevents unrealistic high flow velocities near the contact points. Regarding the calculated pressure drop, both methods show good results and are in good agreement with correlations and experimental values by $\pm 10\%$ (Ookawara et al. 2007, Eppinger et al. 2011, Wehinger et al. 2015a). For non-spherical particles, the situation is more complex, as contact points, lines, and areas occur. Wehinger et al. (2017a) investigated the effects of contact modifications in a bed of cylinders by using the caps and bridges method for line and area contacts and caps and united method for overlaps resulting from composite DEM particles. The proposed method detects the different contact modes in a packed bed and modifies them locally. The bridges method only works for cylinder-like particles

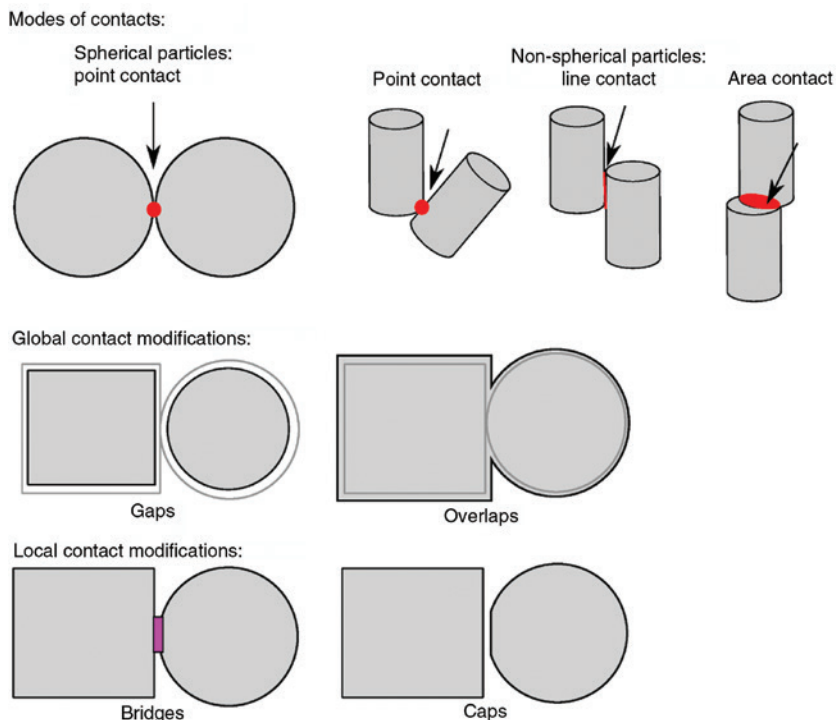


Figure 6: Modes of contacts in packed beds (top) and modification strategies (bottom). Reprinted with permission from Wehinger et al. (2016b). Copyright (2016) American Chemical Society.

including those with internal voids as an algebraic relation is needed to determine the type of contact. The local caps method is more flexible and applicable to particles with internal voids [see Eppinger et al. (2014a) and Wehinger et al. (2015b)] and shapes that are not cylinder-like as tri-lobes. Independent of the contact type, the meshing algorithm automatically detects the faces that are in proximity to each other and creates a small gap.

If heat transfer is incorporated, both approaches show significant differences compared to each other. Figure 24 offers an overview of the different heat transfer mechanisms in a fixed bed of particles. It is obvious that the local caps and the local bridges method will lead to different results regarding the particle-particle and particle-wall heat transfer by conduction through the contacts. By introducing the small space at the contact, the local caps approach neglects the inter-particle conduction. Nevertheless, it was shown by Slavin et al. (2000, 2002) that heat transfer by inter-particle conduction can be neglected, if particles show non-plastic behavior and are not compressed. The numerical heat transfer study of Eppinger et al. (2014b) and Wehinger et al. (2016b) showed that the local caps meshing approach leads to results that show good agreement with experimental data concerning radial and axial temperature profiles. As reported

by Dixon et al. (2013b) and Wehinger et al. (2017a), the local bridges approach leads to an overestimation of the radial heat transfer if the thermal conductivity of the bridges is not adapted to replace the original particle-fluid-particle heat transfer. As a difficulty remains the choice of the thermal conductivity of these bridges. For spherical particles, Dixon et al. (2013b) developed a relationship to calculate the effective thermal conductivity of the bridges. However, for non-spherical particles, this is still a matter of on-going research. It might be attractive to use this variable as fitting parameter, but this should definitely be avoided as it contradicts the philosophy of first principle modeling. Furthermore, the diameter of the bridges can be a crucial parameter. Dixon et al. (2013b) investigated the impact this parameter has on the pressure drop and heat transfer. They found that for flow or pressure drop, the bridge diameter should be below 20% of d_p for particle-particle contacts and below 30% of d_p for particle-wall contacts. If heat transfer is taken into account, the bridges should not exceed a diameter of 20% of d_p for $Re_p \leq 2000$ or 10% of d_p for higher Reynolds numbers. Rebughini et al. (2016) studied the impact of the bridge size for reactive CFD simulations of heterogeneous catalytic fixed-bed reactors. They concluded that the conversion is independent of the bridge diameter if

the bridge-to-particle diameter ratio is chosen according to the fluid dynamic recommendation of Dixon et al. (2013b).

As a summary, the modification of the bed shape should be minimized. Hence, local contact modifications should be preferred to global contact modifications. The local bridges method shows good results for pressure drop and heat transfer prediction. Still, the choice of the thermal conductivity of these bridges needs further investigation. Contrarily, the local caps method shows strengths due to its low time consumption, numerical stability, and straightforward parameter selection. Dixon et al. (2012b) directly compared the caps and bridges methods and found for higher flow rates that there was little differences between the two methods, mainly because the heat transfer is dominated by convection in this case. For further clarification, more detailed experiments are needed of heat and mass transfer in low N packed beds with which CFD can be validated adequately.

2.3 Solving governing equations

The fundamental formulations of the governing equations for laminar and turbulent flow and the mathematics for solving those equations have been published broadly in the literature (Ferziger and Peric 1999, Ranade 2002, Kee et al. 2003). This and the following sections are intended to summarize the most important aspects for fixed-bed simulations and make no claim to be complete. The set of governing equations consists of conservation of total mass, conservation of momentum, conservation of mass of chemical species i , and conservation of energy in terms of specific enthalpy. Here, the set of equations is formulated in Cartesian coordinates assuming a laminar problem. For the turbulent formulation, see Section 2.4.

Conservation of total mass:

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{v}) = 0, \quad (4)$$

where ρ is the mass density, t is the time, and \mathbf{v} is the velocity.

Conservation of momentum:

$$\frac{\partial (\rho \mathbf{v})}{\partial t} + \nabla \cdot (\rho \mathbf{v} \mathbf{v}) = \nabla \mathbf{T} + \rho \mathbf{g}, \quad (5)$$

where \mathbf{g} is the gravity vector and the stress tensor \mathbf{T} is written as:

$$\mathbf{T} = -\left(p + \frac{2}{3}\mu \nabla \cdot \mathbf{v}\right)\mathbf{I} + 2\mu \mathbf{D}, \quad (6)$$

where μ is the mixture viscosity and \mathbf{I} is the unit tensor, p is the pressure, and \mathbf{D} is the deformation tensor:

$$\mathbf{D} = \frac{1}{2}[\nabla \mathbf{v} + (\nabla \mathbf{v})^T] \quad (7)$$

Conservation of species i :

$$\frac{\partial (\rho Y_i)}{\partial t} + \nabla \cdot (\rho Y_i \mathbf{v}) + \nabla \cdot \mathbf{j}_i = R_i^{\text{hom}} \quad \text{for } i = 1, \dots, N_g \quad (8)$$

with mass fraction $Y_i = m_i/m$ of species i and total mass m . N_g is the number of gas-phase species. R_i^{hom} is the net rate of production due to homogeneous chemical reactions. The diffusion mass flux of each species \mathbf{j}_i is described by the mixture-average formulation:

$$\mathbf{j}_i = -\rho D_{M,i} \nabla Y_i, \quad (9)$$

where $D_{M,i}$ is the effective diffusivity between species i and the remaining mixture M , which is defined as follows:

$$D_{M,i} = \frac{1 - X_i}{\sum_{j \neq i}^{N_g} X_j / D_{ij}} \quad \text{for } i = 1, \dots, N_g. \quad (10)$$

The binary diffusion coefficients D_{ij} can be obtained through polynomial fits CD-adapco (2014). M_i is the molecular weight of species i , and T , the temperature. The molar fraction X_i can be written as follows:

$$X_i = \frac{1}{\sum_{j=1}^{N_g} \frac{Y_j}{M_j}} \frac{Y_i}{M_i} \quad (11)$$

Conservation of energy in terms of specific enthalpy h :

$$\frac{\partial (\rho h)}{\partial t} + \nabla \cdot (\rho h \mathbf{v}) + p(\nabla \cdot \mathbf{v}) - (\boldsymbol{\tau} : \nabla \mathbf{v}) + \nabla \cdot \dot{\mathbf{q}} = S_h, \quad (12)$$

where $\boldsymbol{\tau}$ is the viscous stress tensor and S_h , a heat source. The diffusive heat transport $\dot{\mathbf{q}}$ is given by:

$$\dot{\mathbf{q}} = -k \nabla T + \sum_{i=1}^{N_g} h_i \mathbf{j}_i \quad (13)$$

with thermal conductivity of the mixture k and mixture specific enthalpy h :

$$h = \sum_{i=1}^{N_g} Y_i h_i(T) \quad (14)$$

as a function of temperature $h_i = h_i(T)$.

Ideal gas can be assumed connecting pressure, temperature, and density to close the governing equations:

$$p = \rho RT \sum_{i=1}^{N_t} \frac{Y_i}{M_i} \quad (15)$$

Here M_i is the molecular weight of species i .

As already mentioned in the previous section, different numerical methods can be used to solve the balance equations, namely, FEM, LBM, and FVM. All have their benefits and disadvantages that make them more or less applicable for the simulation of fixed-bed reactors.

2.3.1 Finite-element method

Although FEM is rarely used for the simulation of fixed-bed reactors – to the authors knowledge, within the last decade, only Motlagh and Hashemabadi (2008) used the method for the simulation of fluid dynamic and heat transfer study of a very small packed bed ($N=2$) containing 10 cylindrical particles – a brief introduction of that method will be given.

FEM is a very flexible numerical method that is widely used for computational solid and structural mechanics (CSM) but can also be applied to fluid dynamic applications. The idea behind FEM is to divide a body or fluid domain into a number of finite elements that are interconnected at nodal points. The result of FEM is a continuous function that is composed by numerous shape functions, each describing the behavior of the system in one element. The shape functions can have an arbitrary definition, but most often, linear or polynomial functions are used. The solution of the variables is stored in the nodal points.

Using the Galerkin formulation of FEM and assuming that the partial differential equation can be written using a differential operator L :

$$L(\phi) = f \quad (16)$$

and the solution ϕ can be estimated by a value $\hat{\phi}$ by a linear combination of several shape functions θ :

$$\hat{\phi} = \sum_{k=1}^N b_k \theta_k. \quad (17)$$

Here b_k is a set of free parameters that are used to minimize the residual

$$r = L(\hat{\phi}) - f = L\left(\sum_{k=1}^N b_k \theta_k\right) - f. \quad (18)$$

If the weighted residual method is used, a further restriction is that the overall integral of the residual multiplied with a test function should vanish. This is called the weak formulation:

$$\int_{\Omega} r \Psi d\Omega = \int_{\Omega} \left(L\left(\sum_{k=1}^N b_k \theta_k\right) - f \right) \Psi d\Omega = 0. \quad (19)$$

The equation includes N unknown values for b_k . Therefore, N different and linear independent test functions Ψ_k are needed to derive an equation system containing N equations. A widely used approach is to set:

$$\Psi_k = \theta_k. \quad (20)$$

The advantage of FEM is that it provides a rather generalized framework for the solution of arbitrary problems. It is also known to show low numerical diffusion and is therefore suited for highly viscous or visco-elastic flow problems. On the other hand, FEM is a very memory-demanding method, which limits its usage.

2.3.2 Finite-volume method

FVM is a numerical approach to solve the governing equations by the discretization of the numerical domain into a number of finite volumes, often called cells. The system of partial differential equations is integrated over each element and, by this, transferred into a system of algebraic equations that can be solved. The governing equation of a conserved quantity in its integral form can be written as follows:

$$\oint_S \rho \phi \mathbf{v} \cdot \mathbf{n} dS = \oint_S \Gamma \nabla \phi \cdot \mathbf{n} dS + \int_V q_\phi dV. \quad (21)$$

Equation (21) is valid for each cell and, therefore, the overall domain. By its definition, FVM is an inherently conservative numerical method. To gain an algebraic expression for each cell, the area and volume integrals in Eq. (21) need to be approximated.

The overall flux across the cell faces can be expressed as the sum of fluxes over each cell face k :

$$\oint_S f dS = \sum_k \oint_{S_k} f dS, \quad (22)$$

where f represents the convective ($\rho \phi \mathbf{v} \cdot \mathbf{n}$) or diffusive ($\Gamma \nabla \phi \cdot \mathbf{n}$) flux in the direction of the face normal. Figure 7 shows schematically a two-dimensional (2D) Cartesian mesh. It is common to use the compass notation to identify the centers of the considered cell and its neighbors (P, N, S, W, E), the faces (n, s, w, e), and the vertices (nw, ne, se, sw).

The simplest approximation for the surface integral of the eastern face is given by the assumption that the mean face value of the variable is equal to its value in the center of the cell face:

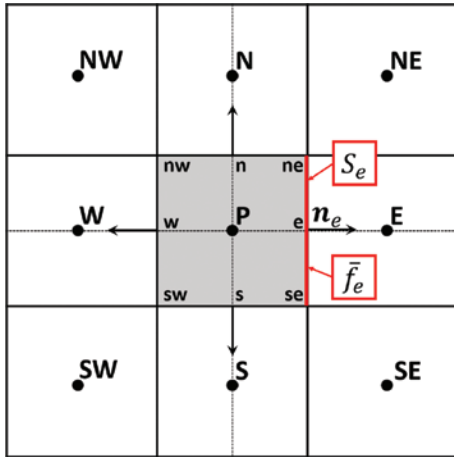


Figure 7: Mesh topology and notation on a 2D Cartesian mesh.

$$\oint_{S_e} f dS = \bar{f}_e S_e \approx f_e S_e \quad (23)$$

More complex approximations, leading to higher order methods, like the Trapezoidal rule and the Simpson's rule – which take into account information from neighboring cells – are also possible but are discussed elsewhere [see Ferziger and Peric (1999)].

To estimate the volume integral in Eq. (21), one approach can be to assume that the value stored in the cell center equals the mean value in the control volume:

$$\int_V q_\phi dV = \bar{q}_\phi \Delta V \approx q_{\phi,P} \Delta V \quad (24)$$

Obviously, for the calculation of the convective and diffusive fluxes, variable values at the face centers are needed. Different methods exist to approximate the face values based on the cell values (e.g. upwind and central-differencing), which are extensively discussed by Ferziger and Peric (1999). The resulting algebraic equation system can subsequently be solved by applying appropriate numerical methods.

As it can be seen in Table 1, for the majority of the numerical work done in the field of fixed-bed reactors, FV technique is used nowadays. This is, on the one hand, related to its applicability on unstructured meshes and the progress that was made in the last decade regarding automated meshing algorithms. On the other hand, to its beneficial characteristics like noninvasive boundary conditions – as the variables are stored in the cell centers and the boundary condition acts on the surface – and the fact that the FVM conserves mass, energy, and momentum by definition.

Furthermore, the incorporation of additional relevant physical phenomena can easily be done by either solving

additional transport equations (e.g. heat and mass transfer, turbulence), including source terms (e.g. chemical reactions), or by applying special boundary conditions [e.g. conjugated heat transfer (CHT) and surface-to-surface radiation]. It is therefore the ideal framework for multiphysics applications like fixed-bed reactors.

2.3.3 Lattice-Boltzmann method

LBM is compared to FVM and FEM not only as a different method to solve the system of partial differential equations. The underlying physical perspective is completely different by using the kinetic theory of gases. The basis of this approach is the assumption that continuous mechanical phenomena are the result of statistical averaged effects on a molecular level. As not all molecules and interactions can be taken into account because of the immense numerical effort this would cause, only a limited number of representative particles are taken that are allowed to move on a discrete lattice. The exchange of momentum and energy is achieved by a sequential collision step followed by the motion of the particles along the lattice – often called streaming.

For a 2D simulation, each particle has nine possible directions to move, including the possibility to rest. Associated with each direction is a so-called microscopic velocity \mathbf{e}_i depicted in Figure 8A, where $i=0 \dots 8$. For each direction, also a probability f_i exists that the particle moves in this direction. During the collision, step rules are applied that need to conserve mass, momentum, and energy. This is done by applying a collision term Ω to the following equation:

$$f_i^*(\mathbf{x}, t) = f_i(\mathbf{x}, t) + \Omega \quad (25)$$

Using the Bhatnagar-Gross-Krook (BGK) operator, the collision term Ω can be expressed as follows:

$$\Omega = \frac{f_i - f_i^{\text{eq}}}{T\omega}, \quad (26)$$

where ω is the relaxation time that is related to the kinematic fluid viscosity ν and f_i^{eq} is a local equilibrium distribution in the direction of \mathbf{e}_i (see Figure 8B).

$$f_i(\mathbf{x}_i + c\mathbf{e}_i\Delta t, t + \Delta t) = f_i^*(\mathbf{x}, t), \quad (27)$$

where c is the lattice speed $c = \frac{\Delta x}{\Delta t}$. A more comprehensive discussion about the use of LBM in the field of CFD can be found in the study by Succi (2001).

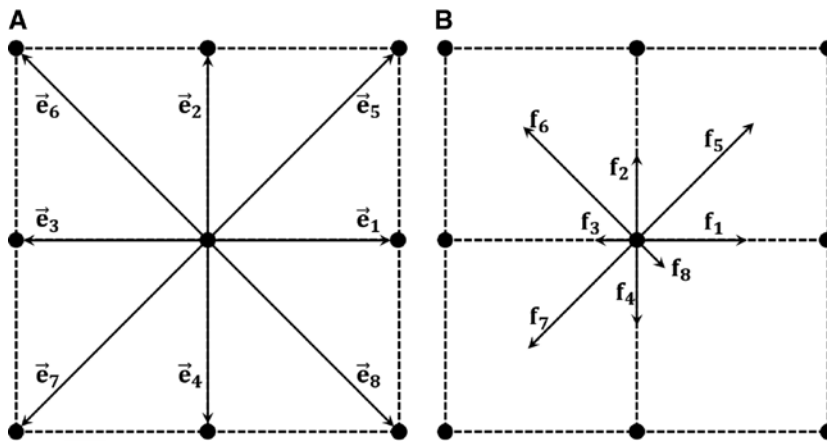


Figure 8: D2Q9 LBM.

(A) Microscopic velocities and (B) weighted by distribution function.

In the early years, LBM has been extensively used by several authors like Zeiser et al. (2001, 2002), Freund et al. (2003, 2005), and Manjhi et al. (2006) for their flow and mass transfer simulations. Compared to other numerical method in these days, LBM had the benefit that the meshing process was very efficient and that the contact point problem could easily be avoided. The results presented by Freund et al. (2003, 2005) showed that LBM is able to predict the pressure drop and the local flow field with a satisfactory accuracy. Compared to correlation, the calculated pressure drop showed a deviation of below $\pm 10\%$ and also the local velocity was in good agreement with Laser Doppler anemometry (LDA) data by Krischke (2001).

During the last decade, LBM got more and more replaced by FVM. This was driven not only by the innovations in the field of FVM, like major improvements in automated meshing and increasing computational power, but also due to limitations of the LBM; as mentioned by Nijemeisland and Dixon (2004) and Freund et al. (2005), the main limitation of LBM is that simulations including CHT are almost impossible to do with LBM as either a more complex discretizations of the velocity space is needed or a separate distribution function for the temperature needs to be incorporated. Both possibilities increase the numerical effort and tend to promote instabilities. Furthermore, as mentioned by Zeiser et al. (2002), LBM is an inherently transient numerical method, which makes its application on steady-state problems less efficient.

To the knowledge of the authors, only the work of Caulkin et al. (2008, 2012) used LBM within the last decade to show the possibility of running fluid dynamic

simulations of non-spherical particles like cylinders, trilobes, and Raschig rings.

2.4 Turbulence

Turbulence is characterized by strongly fluctuating 3D and unsteady eddies. It has a significant impact on the lateral mixing of all transport properties. The flow gets turbulent if the inertial forces become dominant compared to the viscous forces. The transition from laminar to turbulent flow does not happen instantly. Most often, a transition zone exists that can be characterized by a set of critical Reynolds numbers whose values depend on the physical system itself. According to Dybbs and Edwards (1984), the flow regime in fixed-bed reactors can be characterized using the Reynolds number based on the particle diameter and the interstitial velocity Re_ϵ as follows:

1. $Re_\epsilon < 1$: Viscous flow regime. Pressure drop is a linear function of the interstitial velocity.
2. $10 \leq Re_\epsilon \leq 150$: Steady laminar inertial regime. Pressure drop is a non-linear function of the interstitial velocity, and boundary layers are forming.
3. $150 \leq Re_\epsilon \leq 300$: Unsteady laminar inertial regime. The flow shows oscillating behavior in the wake within the voids. At $Re_\epsilon = 250$, laminar vortices start to form.
4. $Re_\epsilon > 300$: Turbulent flow. Characterized by an unsteady and chaotic flow.

Various numerical methods exist to describe turbulent effects in CFD. They differ in the amount of subgrid modeling that is done to account for turbulent eddies. The

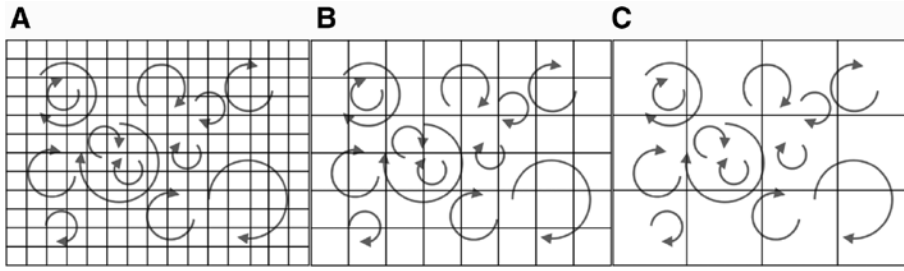


Figure 9: Turbulence models.
(A) DNS. (B) LES and (C) RANS.

three main classes are depicted in Figure 9: DNS, LES, and RANS modeling.

2.4.1 DNS

DNS uses no subgrid modeling to account for turbulent eddies. The turbulent vortices of all length scales, down to the Kolmogoroff scale, are directly mesh-resolved (see Figure 9A), and the Navier-Stokes equations can be used without any modification. Although this approach has the benefit that no further modeling is needed, it is not applicable for most applications as that kind of simulations is inevitably transient and a very fine mesh is needed. Therefore, the numerical effort is excessively high. Shams et al. (2013a,b) investigated the fluid flow and heat transfer in an SC, FCC unit cell arrangement using DNS for $Re_p \approx 3100$. They used the synthetic eddy method to initialize the turbulent eddies in the domain. Although the flow domain was very small, the authors needed around 15 million volume cells to meet the DNS requirement.

2.4.2 LES

For LES, the mesh requirements are less strict than for DNS. The idea behind LES is to resolve large turbulent eddies while the smaller ones are treated by a subgrid model (see Figure 9B). The user can choose the length scale below which the eddies are modeled by applying a filter function. The only obvious requirement is that the directly solved eddies need to be larger than the cell size. Using a filter function $G(\mathbf{x}, \mathbf{x}')$, a filtered transport property $\hat{\phi}$ can be calculated as follows:

$$\hat{\phi}(\mathbf{x}) = \int G(\mathbf{x}, \mathbf{x}') \phi(\mathbf{x}') d\mathbf{x}', \quad (28)$$

where \mathbf{x} is the position vector at the point of interest and \mathbf{x}' loops through all cells in the neighborhood. G determines

the impact of ϕ at \mathbf{x}' on $\hat{\phi}$ at \mathbf{x} . If the distance between \mathbf{x} and \mathbf{x}' is smaller than the filter size, $\phi(\mathbf{x}')$ is taken into account for the calculation of $\hat{\phi}(\mathbf{x})$, otherwise, not. The transport property is combined by filtered term $\hat{\phi}$ and a residual part ϕ^* :

$$\phi = \hat{\phi} + \phi^* \quad (29)$$

The governing equations of LES are obtained by filtering the conservation equations. The filtered momentum balance can be written as follows:

$$\rho \frac{\partial \hat{\mathbf{v}}}{\partial t} = -\rho \hat{\mathbf{v}} \nabla \hat{\mathbf{v}} - \nabla \cdot \hat{\boldsymbol{\tau}}^s + \mu \nabla^2 \hat{\mathbf{v}} - \nabla \hat{p} + \rho \mathbf{g} \quad (30)$$

The subgrid-scale Reynolds stress tensor $\boldsymbol{\tau}^s$ is defined as follows:

$$\tau_{ij}^s = -\rho (\widehat{v_i v_j} - \hat{v}_i \hat{v}_j) \quad (31)$$

Several subgrid-scale models (SGS) exist to describe $\boldsymbol{\tau}^s$. A detailed discussion of those would extend the scope of this work. Therefore, interested readers are recommended to check the applicable literature, e.g. Ferziger and Peric (1999).

Although the mesh requirements for LES-based simulations are much lower compared to DNS, it is still numerically quite demanding – i.e. caused by its inevitably transient nature – and therefore not established in the field of fixed-bed reactors. Shams et al. (2013b) used LES for fluid dynamic and heat transfer simulations in a SC FCC unit cell. They observed good agreement regarding the mean and root-mean-square (RMS) temperature and velocity field between LES and DNS results while saving simulation time by a factor of six. They needed around 6 million volume cells to discretize the flow domain. This corresponds to about one-third of the number of cells needed for DNS but is still too much for the application in extended beds. Later on, Shams et al. (2015) investigated fluid dynamics and heat transfer in a rectangular cut of a

random packing consisting of around 20 spherical particles using LES. The resulting mesh consisted of approximately 18 million volume cells. It took them 6 months to reach statistical convergence on HPC using 120 processors (2.66 GHz). This truly indicates that there are some miles left to go to establish LES in the field of fixed-bed reactors.

2.4.3 RANS

In the majority of academic and industrial applications, RANS turbulence models are used. RANS models do not resolve the turbulent eddies; instead, they are modeled via subgrid model (see Figure 9C). This leads to a tremendous reduction of the numerical effort as there is no more need to run a transient simulation and the cell count can be decreased significantly. Fluid dynamic simulation using RANS models can be conducted within hours on a local workstation for fixed beds with a few thousand of particles. The idea behind that class of turbulence model is that a property can be decomposed into its time-averaged value and a fluctuating component:

$$\phi(x_i, t) = \bar{\phi}(x_i) + \phi'(x_i, t) \quad (32)$$

If Eq. (32) is applied to the governing equations, one gets the RANS equations:

$$\nabla(\rho\bar{\mathbf{v}}) = 0 \quad (33)$$

$$\rho \frac{\partial \bar{\mathbf{v}}}{\partial t} = -\rho \bar{\mathbf{v}} \nabla \bar{\mathbf{v}} - \nabla \bar{\boldsymbol{\tau}}_t + \mu \nabla^2 \bar{\mathbf{v}} - \nabla \bar{p} - \rho \bar{\mathbf{g}}, \quad (34)$$

where $\boldsymbol{\tau}_t$ is the Reynolds stress tensor:

$$\boldsymbol{\tau}_t = \rho \mathbf{v}' \times \mathbf{v}' = \begin{pmatrix} \overline{\rho v'_x v'_x} & \overline{\rho v'_x v'_y} & \overline{\rho v'_x v'_z} \\ \overline{\rho v'_y v'_x} & \overline{\rho v'_y v'_y} & \overline{\rho v'_y v'_z} \\ \overline{\rho v'_z v'_x} & \overline{\rho v'_z v'_y} & \overline{\rho v'_z v'_z} \end{pmatrix} \quad (35)$$

The components of $\boldsymbol{\tau}_t$ need to be determined to close the equation system. For this, different approaches have been developed in the past, such as the Reynolds stress model (RSM) or eddy viscosity models. The main difference between both approaches is that the eddy viscosity models assume isotropic turbulence, while RSM can model anisotropic turbulent behavior. Nevertheless, the class of eddy viscosity models is heavily used in academia and the industry and will therefore be discussed further.

To describe the Reynolds stress tensor, the Boussinesq approximation is used:

$$\boldsymbol{\tau}_t = \mu_t [\nabla \mathbf{v} + (\nabla \mathbf{v})^T] - \frac{2}{3} \rho k \delta, \quad (36)$$

where μ_t is the turbulent viscosity and k is the turbulent kinetic energy, defined as follows:

$$k = \frac{1}{2} |\bar{\mathbf{v}}'|^2 \quad (37)$$

To close the system of equations, an expression for μ_t is needed. For this reason, different models were developed in the past that can be categorized in zero-, one- and two-equation turbulence models. For the simulation of fixed-bed reactors, almost entirely the class of two-equation models has been used in the past and is therefore of interest. This class can be divided into the group of $k-\varepsilon$ and $k-\omega$ turbulence models. Depending on which group is used, μ_t can be expressed as follows:

$$\mu_t = C_\mu \rho \frac{k^2}{\varepsilon} \quad (38)$$

or

$$\mu_t = \rho \frac{k}{\omega}. \quad (39)$$

Here, ε is the turbulent dissipation rate, and ω , the specific dissipation. A great number of models exist to determine k and ε , respectively, ω by solving transport equation for each parameter. In the diffusion term of the energy and mass conservation equation, an additional turbulent thermal conductivity and diffusion coefficient is introduced whose values are correlated with the turbulent viscosity by using a specified turbulent Schmidt $Sc_t = \frac{\mu_t}{\rho D_t}$ and Prandtl number $Pr_t = \frac{\mu_t c_p}{\lambda_t}$. As a detailed discussion of the several models does not fall within the scope of this work, the interested reader is referred to the relevant literature, e.g. Wilcox (2006).

Various authors tested different RANS turbulence models and investigated their applicability to the field of fixed-bed reactors. Coussirat et al. (2007) compared RSM against the Standard $k-\varepsilon$ and the Spalart-Allmaras one-equation model regarding pressure drop and particle Nusselt number in a bed of 44 spheres. The authors validated their numerical results against correlation data and showed that, if a reasonable mesh resolution is used, all tested turbulence models achieve similar satisfying results concerning the particle Nusselt number but the Spalart-Allmaras model tends to under-predict the pressure drop for a wide range of Reynolds numbers. Lee et al. (2007) and Dixon et al. (2011) investigated the heat transfer for a single particle. The former authors tested LES, RSM, and several eddy viscosity models and found that compared to LES, the $k-\omega$ model performs best and produces comparable results, while the latter recommends the use of

the $k - \omega - SST$ turbulence model as it predicts drag coefficient, Nusselt number, and particle temperature reasonable well. Most authors use $k - \varepsilon$ models and its derivatives and achieve very good results even for simulations including heat transfer, mass transfer, or chemical reactions [see Eppinger et al. (2014b), Wehinger et al. (2015a, 2016b, 2017a), and Dixon et al. (2012b)].

2.5 Modeling of chemical surface reactions

Most of the reacting systems, which are realized with fixed-bed reactors, involve predominantly reactions occurring only at the surface of the catalytic pellets. As illustrated in Figure 10, physical as well as chemical processes take place. The fundamental chemical processes in a heterogeneous reaction system can be described with different extent. Detailed surface reaction mechanisms are so-called microkinetic models. Those models indicate the use of a detailed reaction mechanism describing elementary-like processes happening on a catalyst (Saliccioli et al. 2011). Physical phenomena occurring in the pores of the

catalyst pellet (pore diffusion) or through the film around the pellet (film diffusion) together with microkinetics are lumped into so-called macrokinetics. In the following, the fundamentals of modeling chemical surface reactions and their surrounding are summarized briefly. The interested reader is referred to Cortright and Dumesic (2001), Kee et al. (2003), Bird et al. (2007), and Deutschmann (2008).

2.5.1 Description of heterogeneous catalysis

Adsorption processes can be distinguished between physisorption and chemisorption. Physisorption is characterized by weak Van der Waals forces between adsorbate and surface (8–30 kJ/mol). Chemisorption leads to a chemical bonding between adsorbate and surface, which is characterized by high adsorption enthalpies (40–800 kJ/mol) (Kee et al. 2003). The high bonding energy of the adsorbed molecule can lead to dissociation of the molecule, see Figure 11.

Besides measuring rate constants for adsorption processes, collision theory can be applied in terms of gas-phase molecules striking the surface per unit area per unit time F_i (Cortright and Dumesic 2001):

$$F_i = \frac{p_i}{\sqrt{2\pi M_i k_B T}}, \quad (40)$$

where k_B is the Boltzmann constant and p_i is the partial pressure of species i .

The rate of adsorption can then be expressed by multiplying F_i with the sticking coefficient S_i , i.e. the probability that collision with the surface is accompanied with adsorption:

$$\dot{S}_i^{\text{ads}} = F_i \cdot S_i \quad (41)$$

As the sticking coefficient depends on surface coverage Θ and temperature T , it can be defined as the product

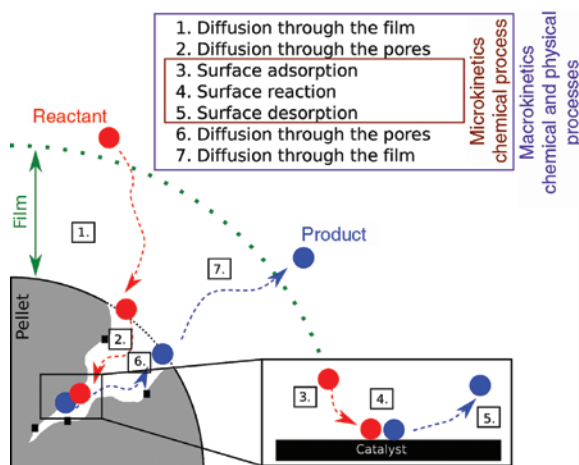


Figure 10: Physical and chemical processes at a catalytic pellet.

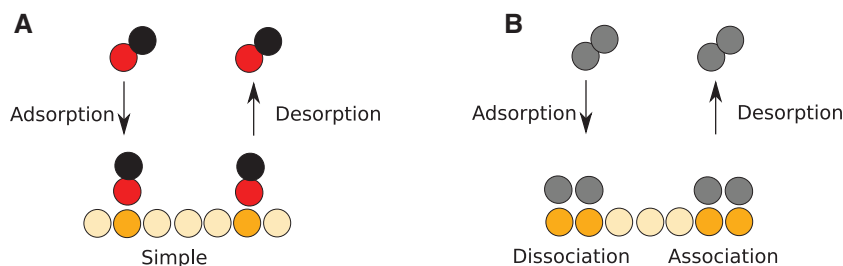


Figure 11: Two adsorption mechanisms shown here; simple and dissociative.

(A) Simple adsorption and desorption. (B) Dissociative adsorption and associative desorption. Reprinted with permission from Wehinger (2016).

of its initial value S_i^0 , i.e. on a clean surface, and the surface coverage (Cortright and Dumesic 2001). The resulting expression for the rate of adsorption is as follows:

$$\dot{S}_i^{\text{ads}} = S_i^0 \sqrt{\frac{RT}{2\pi M_i}} c_i \prod_{j=1}^{N_i} \Theta_j \quad (42)$$

with c_i is the molar concentration.

Reactions between or with adsorbates can be expressed by two different mechanisms, i.e. Langmuir-Hinshelwood and Eley-Rideal, see Figure 12. The Langmuir-Hinshelwood mechanism assumes that both reactants are adsorbed at the catalytic surface. On the other hand, the Eley-Rideal mechanism describes the reaction between one gas phase molecule and a surface adsorbed species.

A first approximation of the pre-exponential factor of any elementary reaction can be assumed to be $10^{13} N_A / \Gamma$ [$\text{cm}^2 / (\text{mol s})$], with N_A being the Avogadro's number, and Γ is the site density of the catalyst. The order of magnitude of $k_B T / h$ is 10^{13} [1/s], h being Planck's constant. In Figure 13A,

a schematic of thermodynamic property as function of reaction coordinate is illustrated. Pre-exponential factors can be calculated from transition state theory (TST) as a function of temperature. The transition state separates the phase space (the space of atomic coordinates and momenta) into a reactants region and a products region with a “dividing surface” orthogonal to the reaction coordinate (Fernández-Ramos et al. 2006). TST expresses rate constants with the Gibbs free energy G of reactants, products, and transition states (Saliccioli et al. 2011):

$$k_i = \frac{k_B T}{h} \exp\left(\frac{-\Delta G_i^\ddagger}{k_B T}\right) = \frac{k_B T}{h} \exp\left(\frac{-\Delta S_i^\ddagger}{k_B T}\right) \exp\left(\frac{-\Delta H_i^\ddagger}{k_B T}\right) \quad (43)$$

$$K_i = \exp\left(\frac{-\Delta G_{i,\text{rxn}}}{k_B T}\right) = \exp\left(\frac{-\Delta S_{i,\text{rxn}}}{k_B T}\right) \exp\left(\frac{-\Delta H_{i,\text{rxn}}}{k_B T}\right) \quad (44)$$

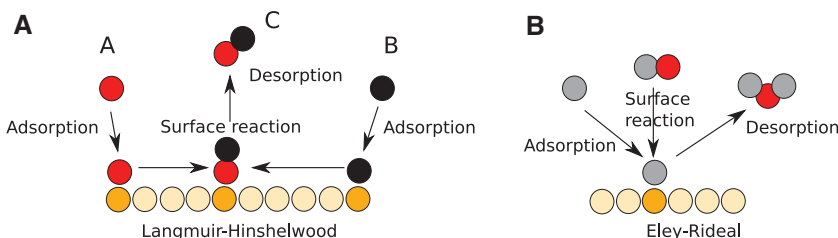


Figure 12: The two mechanisms illustrated in a simplified manner.

(A) Langmuir-Hinshelwood and (B) Eley-Rideal mechanism. Reprinted with permission from Wehinger (2016).

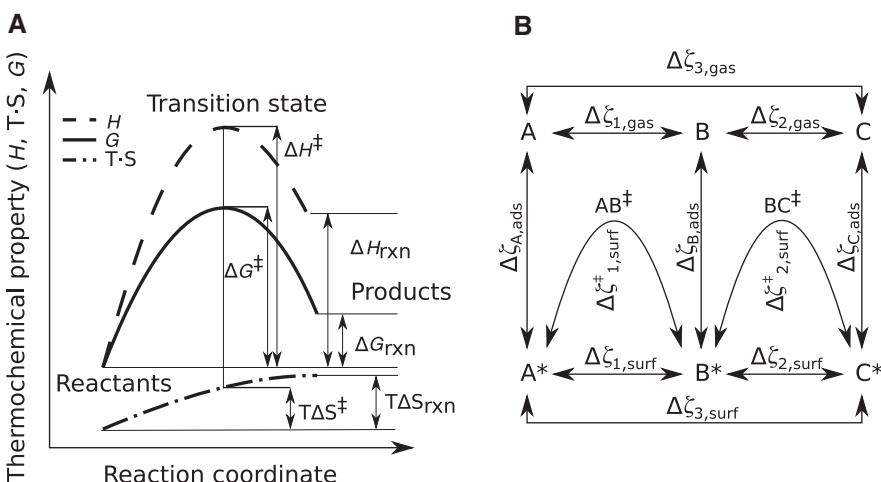


Figure 13: Scheme of thermodynamic property as function of reaction coordinate (A). Diagram of thermochemical property changes $\Delta\zeta$ in model $A \rightleftharpoons B \rightleftharpoons C$ surface reaction mechanism (B).

Partly adopted from Saliccioli et al. (2011). Reprinted with permission from Wehinger (2016).

In the two equations above, k_i describes the irreversible elementary reaction rate constant of reaction i in dependency of the change in Gibbs free energy to transition state ΔG_i^\ddagger , the change in entropy ΔS_i^\ddagger , as well as the change in enthalpy ΔH_i^\ddagger from reactant to transition state. Eq. (44) relates the equilibrium constant K_i of reaction i to the change in free energy of reaction and entropy and enthalpy of reaction.

2.5.2 Modeling rates of heterogeneous catalysis

In principle, heterogeneously catalyzed gas-phase reactions can be described entirely by the sequence of elementary reaction steps of the catalytic cycle consisting of adsorption, surface reaction, and desorption, as described in the above section. However, the level of detail can differ from macroscopic description (power-law kinetics) to the molecular level [density functional theory (DFT)]. In Table 2, the most common methods of modeling rates of heterogeneous catalysis are summarized.

2.5.2.1 Power-law kinetics

In the past, the power-law functional form was the usual type of rate expression:

$$\dot{s} = k_{\text{eff}} C_A^a C_B^b, \text{ with } k_{\text{eff}} = A_{\text{eff}} e^{-E_{\text{eff}}/RT}. \tag{45}$$

In power-law kinetics, the molar net production rate \dot{s} is estimated by an effective rate constant (k_{eff}), reaction order (a, b), as well as an effective activation energy (E_{eff}). Although this type of kinetics is represented by fundamental limitations and a lack of predictive order, it is still commonly used in reactor and process design applications. The reason is the small amount of parameters that have to be regressed to a limited experimental data set.

2.5.2.2 Langmuir-Hinshelwood-Hougen-Watson (LHHW) kinetics

For many years, LHHW kinetics had been a popular simplified approach to describe heterogeneous catalysis in technical reactors. Developing a LHHW kinetics starts with a detailed reaction mechanism. In the following step, *a priori* assumptions are made about fast and slow reaction steps. In general, one rate-determining step (RDS) is identified and it is assumed that adsorption-desorption processes of reactants and products are in partial equilibrium (PE). Surface coverages are referred to partial pressures in the gas phase by means of Langmuir adsorption isotherms.

Table 2: Methods of modeling rates of heterogeneous catalysis.

Modeling method	Simplifications	Application
<i>Ab initio</i> calculation	Most fundamental approach	Simple chemical systems
DFT	Replacement of the N electron wave function by the electron density	Dynamics of reactions, activation barriers, adsorbed structures, frequencies
(Kinetic) Monte Carlo	Neglect of details of dynamics	Adsorbate-adsorbate interactions on catalytic surfaces and nanoparticles
Mean-field approximation	Neglect of details on adsorbate-adsorbate interactions	Microkinetic modeling of catalytic reactions in technical systems
LHHW kinetics	Need of rate-determining step	Modeling of catalytic reactions in technical systems
Power-law kinetics	Neglect of all mechanistic aspects	Scale-up and reactor design for black-box systems

Adopted from Kunz et al. (2012).

The kinetic parameters are determined by fitting the rate equations to (a limited number of) experimental data. Due to this procedure, multiple rate expressions can describe the same set of data with similar statistics, i.e. rate expression multiplicity (Prasad et al. 2009). Moreover, multiple parameter values, describing experimental data reasonably well, may be present for the same rate expression, i.e. rate constant multiplicity. Assuming that the rate expression reproduces the data reasonably accurate, the physics might be wrong, i.e. wrong RDS and PE, or the parameters are physically irrelevant. In addition, LHHW kinetics is typically restricted to one smaller range of operating conditions where the rate changes monotonically regarding one parameter. Saliccioli et al. (2011) compared different values of heats of adsorption from LHHW kinetics and from DFT or experiments. In most cases, the parameters from LHHW kinetics were physically unrealistic, even though the kinetics describes the experimental data fairly well in the investigated range. It can be concluded that with LHHW kinetics, it is possible to reproduce experimental data, but the fundamental mechanism might still be undesignated.

2.5.2.3 Mean-field approximation

The catalytic processes at the reacting surface occur at much smaller time and length scales as the surrounding flow field. An efficient model coupling CFD and microkinetics is the mean-field approximation. On the other hand, there are recent attempts to couple CFD with the computationally expensive kinetic Monte Carlo simulations (Majumder and Broadbelt 2006, Matera and Reuter 2009, Schaefer and Jansen 2013, Matera et al. 2014). The discussion of kMC is out of scope of this review. The interested reader is referred to Sabbe et al. (2012) and Schaefer and Jansen (2013). The mean-field approximation model assumes uniformly distributed adsorbates and catalytic sites over a computational cell. Spatially localized effects, i.e. surface facets, surface defects, and coverage effects, as well as interactions between adsorbates, are neglected by using averaged values. The condition of the catalyst in a computational cell is determined by temperature T and a set of surface coverages Θ_i , which is defined as the fraction of surface covered by species i . Chemical reactions occurring at the catalytic surface are coupled via boundary conditions with gas-phase species concentration at the gas-surface interface. In most of the cases, the catalytically active surface area cannot be resolved. For example, the catalytically active surface of a porous sphere is much larger than its geometric surface. In order to couple the external flow with the surface reactions, the relation

between catalytically active surface area to geometric surface area is needed ($F_{\text{cat/geo}}$). This value can be determined experimentally, for example, by chemisorption measurements. Under steady-state conditions, gas-phase molecules of species i , which are produced/consumed at the catalytic surface by desorption/adsorption, have to diffuse from/to the catalyst (Kee et al. 2003):

$$\mathbf{n}(\mathbf{j}_i) = R_i^{\text{het}} \quad (46)$$

with the outward-pointing unit vector normal to the surface \mathbf{n} and the diffusion mass flux \mathbf{j}_i . The heterogeneous reaction term R_i^{het} can be formulated as follows:

$$R_i^{\text{het}} = F_{\text{cat/geo}} M_i \dot{s}_i \quad (47)$$

with M_i as the molar weight, \dot{s}_i as the molar net production rate of gas-phase species i , and $F_{\text{cat/geo}}$ as the ratio of catalytic active area $A_{\text{catalytic}}$ to geometric area $A_{\text{geometric}}$:

$$F_{\text{cat/geo}} = A_{\text{catalytic}} / A_{\text{geometric}} \quad (48)$$

The molar net production \dot{s}_i results in:

$$\dot{s}_i = \sum_{k=1}^{K_s} \nu_{ik} k_{f_k} \prod_{j=1}^{N_g+N_s} c_j^{\nu'_{jk}}, \quad (49)$$

where K_s is the number of surface reactions, c_j is the species concentrations in [mol/m³] for the adsorbed species N_s and in [mol/m³] for the gas phase species N_g , respectively. In addition, the surface coverage Θ takes into account the surface site density Γ [mol/m²], representing the maximum number of species that can adsorb on a unit surface area. Furthermore, a coordinate number σ_i expresses the number of surface sites, which are covered by this species:

$$\Theta_i = c_i \sigma_i \Gamma^{-1} \quad (50)$$

The time-dependent variation of Θ_i can be written as follows:

$$\frac{\partial \Theta_i}{\partial t} = \frac{\dot{s}_i \sigma_i}{\Gamma} \quad (51)$$

Under steady-state conditions, the left side of Eq. (51) equals zero. The reaction rate expression can be modified by the concentration, or coverage, of some surface species Θ_i (Kee et al. 2003):

$$k_{f_k} = A_k T^{\beta_k} \exp\left(\frac{-E_{a_k}}{RT}\right) \prod_{i=1}^{N_s} 10^{\eta_{ik} \Theta_i} \Theta_i^{\mu_{ik}} \exp\left(\frac{-\epsilon_{ik} \Theta_i}{RT}\right) \quad (52)$$

with three extra coverage parameters, η_{ik} , μ_{ik} , and ϵ_{ik} . The term with η_{ik} represents a change of magnitude of the pre-exponential factor in dependency of surface coverage Θ_i . The term including μ_{ik} indicates the modification of the surface reaction rate expression proportional to any arbitrary power of surface coverage. The ϵ_{ik} term represents a modification of the activation energy as a function of coverage.

The occurrence of adsorption reactions results in a modification of the conventional rate coefficient by referencing sticking coefficients:

$$k_{f_k}^{\text{ads}} = \frac{S_i^0}{\Gamma^\tau} \sqrt{\frac{RT}{2\pi M_i}}, \quad (53)$$

with S_i^0 as the initial (uncovered surface) sticking coefficient and $\tau = \sum_{j=1}^{N_s} \nu'_{jk}$ as the sum of all the surface reactant's stoichiometric coefficients. For more details, see the study by Kee et al. (2003).

2.5.3 Thermodynamic consistency of microkinetics

Thermodynamic consistency of microkinetics is a very important aspect. However, many mechanisms in literature do not prove it explicitly. Thermodynamic constraints of microkinetics can be formulated by four equations (Cortright and Dumesic 2001) proving, firstly, individual elementary reactions and, secondly, the overall net reaction. For individual elementary reactions, the following constraints have to be fulfilled:

$$\Delta E_{i,b} = \Delta E_{i,f} - \Delta H_i \quad (54)$$

$$A_{i,b} = A_{i,f} \exp\left(\frac{\Delta G_i - \Delta H_i}{RT}\right) = A_{i,f} \exp\left(\frac{\Delta S_i}{R}\right), \quad (55)$$

where $\Delta E_{i,b}$ is the backward activation energy, $\Delta E_{i,f}$ is the forward activation energy, and ΔH_i is the standard enthalpy change of the individual reaction step. $A_{i,b}$ is the reverse pre-exponential factor, and $A_{i,f}$ the forward pre-exponential factor, respectively. ΔG_i is the change in standard Gibbs free energy of reaction i , and ΔS_i is the change in entropy.

Applying Hess' law, each reaction step can be formulated with a gas-phase reaction having the same stoichiometry. A net reaction that begins with gaseous reactants and ends with gaseous products can be described as a linear combination of several reactions. Thermodynamic constraints for the net reactions can be defined by the following:

$$\sum_i \sigma_i (\Delta E_{i,f}) - \sum_i \sigma_i (\Delta E_{i,b}) = \Delta H_{\text{net}} \quad (56)$$

and

$$\prod_i \left(\frac{A_{i,f}}{A_{i,b}} \right)^{\sigma_i} = \exp\left(\frac{\Delta G_{\text{net}} - \Delta H_{\text{net}}}{RT}\right), \quad (57)$$

where the subscript "net" denotes the change in thermodynamic state properties from net reactants to net products (Saliccioli et al. 2011). Alternatively, the thermodynamic consistency is defined via the equilibrium constants of each elementary step, as well as the net reaction (Cortright and Dumesic 2001):

$$\prod_i K_{i,\text{eq}}^{\sigma_i} = \prod_i \left(\frac{k_{i,\text{for}}}{k_{i,\text{rev}}} \right)^{\sigma_i} = K_{\text{net}}, \quad (58)$$

where K_{net} is the equilibrium constant of the overall stoichiometric reaction.

In Figure 13B, the relationship is illustrated between gas-phase and surface-phase thermochemical properties of gas and surface intermediates and transition states, denoted as ‡ for the simple $A \rightleftharpoons B \rightleftharpoons C$ surface reaction mechanism. The variable ζ represents any of the thermochemical properties, i.e. free Gibbs energy G , entropy S , and enthalpy H . In the figure, $\Delta \zeta_{3,\text{gas}}$ represents the property change connected with the elementary gas-phase reactions; in this situation, $\Delta \zeta_{3,\text{gas}} = \Delta \zeta_{1,\text{gas}} + \Delta \zeta_{2,\text{gas}}$.

Violation of thermodynamic consistency can lead to erroneous predictions of heat and mass. Enthalpic inconsistency leads to incorrect solution of the energy balance. As a consequence, in a non-isothermal simulation, wrong temperatures are predicted and likewise wrong conversion. In an isothermal simulation, the temperatures are fixed; however, the enthalpic inconsistency leads to errors in the mass balance. On the other hand, inconsistency in terms of entropy is characteristic for a fundamental inconsistency of pre-exponential factors. By defining both the forward and reverse reaction steps, an incorrect equilibrium constant can be the result of thermodynamic inconsistency. This results in an incorrect prediction of the equilibrium state (Mhadeshwar et al. 2003). As a conclusion, guaranteeing consistency of enthalpy and entropy of microkinetics is of fundamental importance.

2.5.4 Solving CFD systems coupled with microkinetics

It is computationally expensive to solve detailed reaction mechanisms in the CFD environment, especially due to

the chemical source terms. In addition, the large number of chemical species, i.e. gas-phase species and surface adsorbed species, and the high number of cells required in complex geometries affect the computational cost dramatically. As a consequence, the full coupled system of microkinetics and CFD has to be avoided. Over the last decades, several different methodologies were developed to reduce the computational cost typical of CFD simulations combined with microkinetic descriptions of the surface chemistry. In the following sections, details of different methodologies are given.

2.5.4.1 The operator-splitting algorithm

The operator-splitting algorithm separates flow field and chemistry time scales.

The generic transport equation of scalar ϕ is given by the following:

$$\frac{\partial(\rho\phi)}{\partial t} + \nabla \cdot (\rho\phi\mathbf{v}) - \nabla \cdot (\Gamma\nabla\phi) = S_\phi, \quad (59)$$

where the first term states the transient change, the second accounts for convection, the third is for diffusion, and the right hand side gives the source term. Γ is the diffusion coefficient.

The time integration of the chemical state (species mass fractions Y_k and enthalpy h) is carried out in two steps:

1. At the beginning of each time step, the chemical state is integrated from state $(Y_k, h)^n$ to $(Y_k, h)^*$, taken only the chemical source terms into account:

$$\begin{aligned} \frac{\partial(\rho\phi)}{\partial t} &= S_\phi \\ \text{for species mass fractions: } \phi &= Y_k, \quad S_\phi = R_i^{\text{het}} \\ \text{for enthalpy: } \phi &= h, \quad S_\phi = S_h^{\text{het}} \end{aligned} \quad (60)$$

The system of Eq. (60) is solved typically with specialized stiff-ODE solvers.

2. The flow field is then integrated from $(Y_k, h)^*$ to $(Y_k, h)^{n+1}$ without the chemical source term. The following system of equations is computed:

$$\begin{aligned} \frac{\partial(\rho\phi)}{\partial t} + \nabla \cdot (\rho\phi\mathbf{v}) - \nabla \cdot (\Gamma\nabla\phi) &= 0 \\ \text{for species mass fractions: } \phi &= Y_k \\ \text{for enthalpy: } \phi &= h \end{aligned} \quad (61)$$

This algorithm is more suitable for transient simulations, although it can also be applied for steady-state simulations. If the problem is steady state, a pseudo-time-step, which is based on convection and diffusion

fluxes in that cell, is introduced to integrate the ODE for the chemistry step. The operator-splitting algorithm is implemented in several commercial CFD software. For example, in STAR-CCM+, an internal add-in code (Dars-CFD Reaction model) is used to solve the stiff ODEs (CD-adapco 2014).

Particle-resolved CFD simulations coupled with detailed reaction mechanisms were realized with STAR-CCM+ for dry reforming of methane in a fixed-bed reactor consisting of spheres, cylinders, and one-hole cylinders (Wehinger et al. 2015a,b, 2016b), as well as in an open-cell foam for the catalytic partial oxidation of methane (Wehinger et al. 2016a). Maestri and Cuoci (2013b) implemented the operator-splitting algorithm in the OpenFOAM environment, called catalyticFOAM (Maestri and Cuoci 2013a).

Hettel et al. (2015) coupled OpenFOAM with DETCHEM (Deutschmann et al. 2014), which allows the integration of detailed surface chemistry into the CFD environment. For a catalytic monolith reactor, the flow regions in the channel were calculated with one-dimensional (1D) or 2D models with DETCHEM. As a conclusion, the operator-splitting algorithm is advantageous for handling the stiffness of the surface chemistry by using specific numerical solvers. However, it does not reduce significantly the computational cost of the simulation.

2.5.4.2 In situ adaptive tabulation (ISAT)

Decades ago, the combustion community was interested in reducing the calculation effort for CFD simulations caused by the large homogenous kinetic models. A dynamic method with a rate tabulation procedure was developed (Maas and Pope 1992). No assumption is needed about reactions in partial equilibrium or about species in steady state. The follow-up of this method is the ISAT method, which can incorporate full homogenous kinetic mechanisms in transient simulations of turbulent flow (Pope 1997). It is reported that the computational time for the full microkinetic reaction scheme was not increased compared with the simplified global kinetics methods. Mazumder (2005) coupled the ISAT algorithm for heterogeneous catalysis with a steady-state reacting flow code. The original ISAT method for homogeneous reactions was developed to solve an initial value problem. However, for heterogeneous reaction calculations, a solution is required for a set of nonlinear algebraic equations at boundary faces/nodes. Kumar and Mazumder (2011) showed speed-up factors of approximately 5–11 for steady-state methane-air combustion on platinum using ISAT. This method was recently extended for accelerating the

simulation of complex heterogeneous chemical kinetics within transient, 3D CFD (Blasi and Kee 2016). The authors used the open-source chemical and thermodynamics package Cantera (Goodwin et al. 2016) to solve the kinetics problem with ISAT handling tabulation and retrieval incorporated via user-defined functions (UDF) in ANSYS Fluent 15.0. As a test case, the authors simulated methane reforming in a catalytic microchannel reactor, taking into account coupled fluid mechanics, catalytic chemistry, and conjugate heat transfer. Speed-up factors of 10–20 were reported, which are 2–3 orders of magnitude smaller than for homogeneous reactions (Blasi and Kee 2016). This is due to the fact that the amount of chemically active cells is lower for heterogeneous reactions than for homogeneous reactions. Remember that the chemically active cells are connected to the catalytic surface. Consequently, the saving potential is lower. Recently, methane steam reforming in a packed-bed reactor was simulated with CFD by applying the ISAT algorithm in the catalyticFOAM framework (Bracconi et al. 2017). The original ISAT method was extended by periodically cleansing and reshaping the ISAT table, removing infrequently used leaves and preserving the efficiency of the ISAT binary tree. A speed-up factor of 15 was realized for an isothermal fixed-bed simulation.

2.5.4.3 Pre-computed chemical reaction rate data

An additional answer to the computational effort challenge is using pre-computed solutions of the chemical rate equations. The basic idea is to solve the chemical reaction steps in a previous step for a wide range of operating conditions. Based on these data, an interpolation function is built. While solving the CFD problem, the interpolation function is called rather than the actual surface kinetic model. Several different interpolation functions were developed over the last two decades: higher order multivariate polynomials (Brad et al. 2005), local linear interpolation in the vicinity of mapped data points (Pope 1997), splitting the input space in subregions, and representing the function on each subregion by a simple linear or polynomial function (Brad et al. 2007), neural networks (Hosseini et al. 2012), multivariate spline functions (Votsmeier et al. 2010), reduced Hermite splines (Klingenberger et al. 2017), which scale down significantly the required memory.

Pre-computed reaction-rate data mapping for a chemical reactor was proposed by Votsmeier (2009). An error less than 5% for the conversion rate was found by using a spline representation based on 7000 data points. Speed-up factors of three orders of magnitude were shown for a 3D model of a monolith channel including the irregularly

shaped washcoat. A similar approach was developed by Partopour and Dixon (2016a) with emphasis on independence from the spline toolbox for the reaction-rate evaluation and speed-up. Ethylene and methanol partial oxidation was tested in a fixed-bed of spheres. It is reported that the computational time was not increased compared with global kinetics methods.

2.5.4.4 Cell-agglomeration (CA) algorithm

Another methodology to reduce computational time is the so-called CA algorithm, which was originally developed for dynamic CFD simulations with detailed homogeneous chemistry (Goldin et al. 2009). CA for steady-state heterogeneous chemistry was recently presented by Rebughini et al. (2017). The CA algorithm binds together the computational cells with similar thermo-chemical composition, thus with similar chemistry source terms. In the above-mentioned case, the computational effort is reduced by reducing the number of adsorbed species to be evaluated, as there is no transport term in the governing equations for adsorbed species. Consequently, computational effort is reduced by decreasing the number of chemistry source terms to be estimated. Rebughini et al. (2017) investigated CA for different microkinetic schemes and different reactor geometries, i.e. tubular reactor and packed-bed reactor with a CPOX microkinetic model. A speed-up factor of approximately two was achieved for a tubular reactor consisting of 800 cells. However, a more remarkable speed-up of approximately 14 was found for the packed-bed reactor incorporating 0.6 million cells. It was reported that the decrease in the computational time did not influence the accuracy of the algorithm (Rebughini et al. 2017).

2.5.4.5 Reduction of detailed reaction mechanisms

A different approach to decrease the computational effort of microkinetics is to reduce the mechanism itself. Chemical reaction-network reduction can be achieved by progressive species reduction with re-parametrization (Jacobsen et al. 2002), element flux analysis (He et al. 2010), integer linear program formulation (Mitsos et al. 2008), principle component analysis (Mhadeshwar and Vlachos 2005, Maestri et al. 2008), as well as reaction-route graphs (Fishtik et al. 2004). However, many approaches involve critical re-parametrization and complex numerical methods. Karst et al. (2015) presented a novel technique where the focus is on the sensitivity of the reaction kinetic model. Specified reaction steps are removed, and their significance for the prediction of the overall system performance is evaluated. The methodology was tested for a C1 microkinetic model describing methane conversion

to syngas on a Rh/Al₂O₃ catalyst. The agreement between reduced and original microkinetic model was very well. Finally, the reduced model was utilized to optimize a methane reformer for the production of hydrogen-rich gas mixture.

2.5.5 Calculation of gas-phase properties

The molecular transport of species, momentum, and energy under typical fixed-bed reactor conditions generally happens in a multicomponent gaseous mixture environment. The characterization of this transport requires a fundamental description of diffusion coefficients, viscosities, thermal conductivities, and thermal diffusion coefficients. Pure species properties can be derived with standard kinetic-theory expressions. However, mixture properties can be calculated with a wide range of possibilities. The full multicomponent formulation shows two main advantages over the simplified mixture formulas, i.e. accuracy and mass conservation. The downside is the computational expense of the full multicomponent formulation. Kee et al. (2003) gave a comprehensive overview of transport coefficients including a discussion of the Chapman-Enskog theory, on which most calculations of transport properties are based. In most of the CFD software, the user can choose on the gas-phase properties. We strongly recommend a detailed description, as oversimplified assumptions can lead to erroneous predictions of transport processes.

Data on thermochemistry of gas-phase species are available from many sources, and mostly, the format is either a tabulated value or a polynomial expression. Thermodynamic polynomial data are used to provide heat capacity, enthalpy, and entropy of a species for a wide range of temperatures formulated typically with a seven-coefficient polynomial curve fit. Several databases are available, e.g. Chemkin thermodynamic database (Kee et al. 1987), the NIST chemical kinetics database (Mallard et al. 1992), or online via <http://kinetics.nist.gov> or http://www.me.berkeley.edu/gri_mech/. Kee et al. (2003) gave an overview of different sources of thermochemistry with an emphasis on combustion chemistry. The formulation follows the NASA chemical equilibrium code (McBride and Gordon 1971).

2.5.6 Modeling transport limitations in porous solid matter

In general, transport limitations in catalytically active pellets can influence the reactor dynamics, light-off, as

well as local and global conversion. Moreover, the size of the catalytic particles, crystal structure and defects, their distribution in the porous pellet or on the substrate, as well as the interaction with the supporting material and deactivation in general also determines the activity of the present catalyst (Ertl 2000). However, this level of detail is not taken into account with the applied mean-field approximation describing the heterogeneous chemical kinetics. If the catalyst is deposited into the porous pellet, most of the active centers of the catalyst lie inside rather than at the surface. Transport limitations occur when the intrinsic rate of diffusion of the species, i.e. the participating reactants or products, inside the porous solid/pellet is slow in comparison to the intrinsic rate of reaction. This fact can decline the conversion and therefore decrease the observed reaction rate. The local effectiveness factor η_L is defined as the ratio of reaction rate inside the pellet and the reaction rate at the surface (Hayes et al. 2004):

$$\eta_L = \frac{\dot{r}_{\text{active}}}{\dot{r}_{\text{surface}}}, \quad (62)$$

whereas the factor is in the range of $0 < \eta_L \leq 1$ under isothermal conditions. Several models are available, approximating pore processes, see, for example, studies by Bartholomew and Farrauto (2006) and Bird et al. (2007). In the following, models relevant for CFD simulations are presented shortly.

2.5.6.1 Instantaneous diffusion

Assuming a diminishing transport limitation, the influence of the thickness of the catalytically active layer, porosity, pore diameter, and particle diameter vanishes. This means that internal mass transport is so fast that none of the species is penetrating into the pellet. In other words, all of the reactions occur at the interface between gas phase and porous pellet, i.e. $\eta_L = 1$ in Eq. (62). No additional term is introduced in Eq. (47). The porous pellet is treated as an impermeable solid material. This assumption is valid, if the catalytically active layer is very thin, the pore diameter is large, the diffusion coefficient is large, or the reaction rate is very slow. Several researchers assumed instantaneous diffusion inside the catalytic pellets in fixed-bed reactors (Maestri and Cuoci 2013b, Wehinger et al. 2015a,b, 2016b).

2.5.6.2 Three-dimensional reaction-diffusion model in porous media

The pellet or the catalytically active layer can be modeled as a 3D porous medium. The Navier-Stokes equations in

porous media can be solved analogously to the equations above, i.e. Eqs. (4–12). However, source terms have to be added. As the velocity at the interface between gas phase and pellet is assumed to be zero, a momentum source term is not added to Eq. (5). Three-dimensional mass transfer in porous media is modeled by modifying the mixture diffusion coefficient $D_{M,i}$. It is replaced by an effective diffusion coefficient $D_{\text{eff},i}$, which takes into account mixture diffusion and Knudsen diffusion (Mladenov et al. 2010):

$$\frac{1}{D_{\text{eff},i}} = \frac{\tau_{\text{CL}}}{\varepsilon_{\text{CL}}} \left(\frac{1}{D_{M,i}} + \frac{1}{D_{\text{Knud},i}} \right), \quad (63)$$

where τ_{CL} is the empirically determined tortuosity, which is defined as the deviation of the pores from an ideal cylinder. The Knudsen diffusion coefficient $D_{\text{Knud},i}$ of species i is defined as follows:

$$D_{\text{Knud},i} = \frac{d_{\text{pore}}}{3} \sqrt{\frac{8RT}{\pi M_i}}, \quad (64)$$

with R as the gas constant and T as the temperature. At atmospheric pressure, Knudsen diffusion occurs, if the mean pore diameters are usually smaller than 100 nm (Baerns et al. 2006).

The dusty gas model describes fluxes inside porous media driven by gradients of concentration and pressure (Mason and Malinauskas 1983, Veldsink et al. 1995). Especially if reactions involving a change in the number of molecules are present, non-uniform pressure profiles in the porous catalyst occur. For a description of the model and a critical discussion, see the studies by Keil (2011) and Kerkhof and Geboers (2005).

The heterogeneous reactions are taken into account as a volumetric mass source term $M_i \gamma \dot{s}_i$ given in kg/m³s and occurring at the right hand side of Eq. (8). The catalyst density is γ , whereas t_{CL} is the thickness of the catalytically active layer:

$$\gamma = \frac{F_{\text{cat/geo}}}{t_{\text{CL}}} \quad (65)$$

Due to the large computational effort required for microkinetics, simplified or lumped kinetics is mostly applied for intraparticle reactions (Behnam et al. 2010, Dixon et al. 2010, 2012b, Taskin et al. 2010, Partopour and Dixon 2016b). More recently, however, Wehinger et al. (2017b) implemented the 3D reaction-diffusion model into a CFD simulation of a single sphere. As a showcase, the catalytic CO oxidation on a sphere with a 100- μm reacting layer was simulated using

a comprehensive microkinetics from the literature (Karakaya and Deutschmann 2013).

2.5.6.3 One-dimensional reaction-diffusion equation

A 1D reaction-diffusion equation takes only the direction normal to the surface into account. It is assumed that concentration gradients in the other directions are much smaller. Hence, the gradients of the species i in normal direction inside the catalytically active layer influence directly the surface reaction rate \dot{s}_i (Hayes et al. 2004). Consequently, the reaction-diffusion equation normal to the surface can be written as follows (Mladenov et al. 2010):

$$\frac{\partial \bar{J}_{i,n}^{\text{CL}}}{\partial n} - \gamma \dot{s}_i = 0 \quad (66)$$

$$\bar{J}_{i,n}^{\text{CL}} = -D_{\text{eff},i} \frac{\partial c_{\text{CL},i}}{\partial n}, \quad (67)$$

with $\bar{J}_{i,n}^{\text{CL}}$ as the normal diffusion molar flux of species i , $c_{\text{CL},i}$ as the molar concentration, and $D_{\text{eff},i}$ as the effective Fick coefficient of species i in the catalytically active layer.

Two boundary conditions are necessary to close the equation system in Eq. (66):

$$c_{\text{CL},i}(n=0) = c_{0,i}; \quad \frac{\partial c_{\text{CL},i}(t_{\text{CL}})}{\partial r} = 0 \quad (68)$$

The first boundary condition implies that the concentration at the gas-phase/pellet interface is the given concentration in the gas phase. The second boundary condition states that the catalytically active layer is thick enough to assume a zero gradient in concentration at the layer/support interface. This model has not been implemented yet into particle-resolved fixed-bed reactor CFD simulations. However, honeycomb-reactor CFD simulations were conducted with the 1D reaction diffusion model, e.g. Mladenov et al. (2010).

2.5.6.4 Effectiveness factor approach

An analytical solution exists for the 1D reaction diffusion [Eq. (66)] for a single species if the following two assumptions are made (Mladenov et al. 2010):

1. The species is consumed and the reaction rate is of first order ($\dot{s} = -k \cdot c$, with k being the rate constant).
2. The diffusion coefficient is constant.

An effective surface reaction rate \dot{s}_{eff} can be obtained by the following:

$$\dot{s}_{\text{eff}} = -D_{\text{eff}} \lambda c_0 \tanh(\lambda t_{\text{CL}}), \quad \text{with } \lambda = \sqrt{\frac{\gamma k}{D_{\text{eff}}}} \quad (69)$$

A maximum reaction rate \dot{s}_{max} assuming no diffusion limitation can be written as follows:

$$\dot{s}_{\text{max}} = -F_{\text{cat/geo}} k c_0 = -\gamma t_{\text{CL}} k c_0 \quad (70)$$

The effectiveness factor η describes the ratio of the effective to the maximum reaction rate:

$$\eta = \frac{\dot{s}_{\text{eff}}}{\dot{s}_{\text{max}}} = \frac{\tanh(\lambda t_{\text{CL}})}{\lambda t_{\text{CL}}} = \frac{\tanh \phi}{\phi} \quad (71)$$

In the equation above, the dimensionless term ϕ is often denoted as *Thiele modulus*. It indicates which process is rate-limiting. When ϕ is small ($\phi < 0.4$), the diffusional resistance is too low to limit the rate of reaction. When ϕ is large ($\phi > 4$), a significant diffusional resistance lowers the observed reaction rate. In multi-species systems, one species that satisfies the two assumptions above has to be chosen. The effectiveness factor is calculated and then multiplied with the reaction rates of all of the species:

$$\dot{s}_{\text{eff},i} = \eta \dot{s}_i \quad (72)$$

However, it has to be kept in mind that the species defining the effectiveness factor can be varied by location in the reactor. For example, the species with the slowest reaction rate is chosen. The effectiveness factor model is zero-dimensional, as only the boundary condition at the gas-phase/pellet interface is manipulated. This is a large simplification in comparison to the 3D or even 1D modeling. The η -approach was used for CFD simulations of stagnation-flow reactors (Karadeniz et al. 2013, 2015, Wehinger et al. 2017b) and for a single catalytic sphere (Wehinger et al. 2017b) but not yet in an entire particle-resolved fixed-bed reactor, as the effectiveness factor method has been used in effective continuum models of fixed beds for approximately 80 years.

3 Applications

The first part of this article was dedicated to the different modeling aspects necessary for a reliable CFD simulation of fixed-bed reactors. It has been shown that many challenges have been tackled in the last decade allowing the conduction of virtual experiments using CFD in at least laboratory-scale dimensions. But what are the new insights that CFD can offer to gain better knowledge of the

system? For what kind of applications can CFD already be used to determine performance and efficiency indicators? Is CFD already a comprehensive design tool for fixed-bed reactors as asked by Wehinger and Kraume (2017)?

A fixed-bed reactor is a complex and interlinked system. Chemical conversion rates depend strongly on mixing characteristics and heat transfer, which are directly linked to the fluid dynamic behavior that is massively influenced by the bed morphology. Consequently, it needs to be proven that all physical phenomena can be described accurately before the question raised above can be answered thoroughly.

Therefore, this section will give a review of several publications dedicated to different aspects of fixed-bed reactor modeling. Furthermore, the accuracy and reliability of the results will be evaluated. New findings that help to improve the overall understanding of this multi-phenomena system will be identified.

3.1 Bed morphology

The key to comprehend the fluid dynamics in fixed-bed reactors is the understanding of the complex bed morphology created by the random arrangement of particles. The simplest model to describe the morphology of a fixed-bed is the pseudo-homogeneous approach where it is assumed that all particles are distributed evenly in the domain. According to this, a constant void fraction is expected in the whole reactor. Therefore, a plug-flow-like behavior with a constant interstitial velocity is assumed. The additional pressure drop can be described by a number of parallel capillary tubes with a constant hydraulic diameter. Tortuosity is taken into account by an adapted friction coefficient. The heat transfer characteristic can be described by an effective thermal conductivity that, in the simplest case, is the mean conductivity of the solid and fluid phase.

Figure 14 shows the axial and radial void fraction distribution of an extended fixed-bed reactor ($N=20$) filled with spherical particles. During the filling process, the first particles tend to build a regular arrangement on the bottom of the container and share point contacts with the bottom plate. Therefore, at $z/d_p=0$, the void fraction has a value of one and decreases to its global minimum that is reached after one particle radius. The subsequent particles prefer a stable position. This is fulfilled if they are placed in the notches of the first particle layer. This quite regular arrangement leads to a next sharp local maximum of the porosity followed by a local minimum. With increasing distance from the bottom of

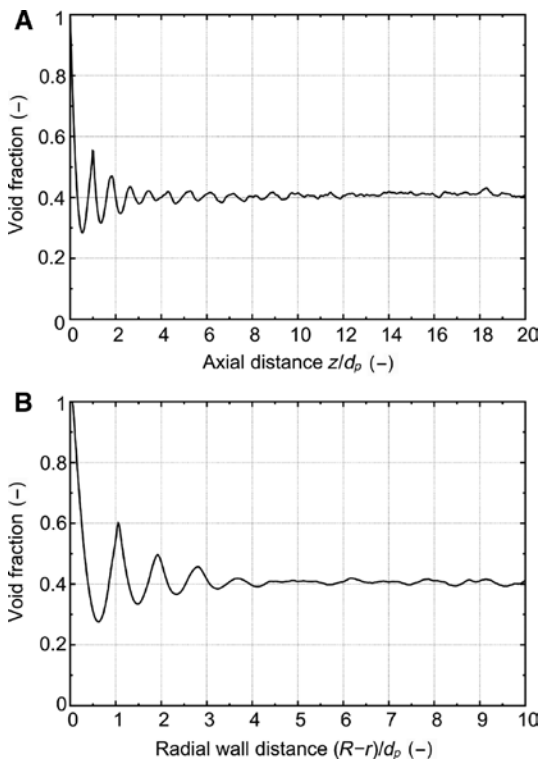


Figure 14: Void fraction profiles for an extended random packing of spheres ($N=20$).
(A) Axial void fraction profile and (B) radial void fraction profile.

the container, the particle arrangement gets more and more random as not all notches of the previous particle layer will be filled. This leads to decreasing oscillations in axial direction until a constant value for the porosity is achieved. After six to eight particle diameters, the axial porosity profile approaches a constant value. In radial direction, the mechanism is similar. The confining walls stabilize the packing. Therefore, the particles next to the wall tend to touch it, as this is the most stable position. As only a point contact is present between particles and wall, the void fraction at the wall has a value of one. At a distance of one particle radius away from the wall, the radial void fraction profile reaches its global minimum as the majority of near-wall particles have their center at this radial position. The second particle layer tries to reach a stable position by arranging in the notches of the first particle layer. This quite regular arrangement leads to a sharp local maximum in porosity at a radial distance of one particle diameter. As not all notches get filled with increasing radial distance, the particle arrangement gets more random with growing spacing to the wall. This leads to declining oscillations of the radial void fraction profile until a constant value is reached after a distance of five particle diameters.

For packed-bed reactors with a low tube-to-particle diameter ratio ($N \leq 15$), the non-homogeneous bed morphology has a significant impact on the fluid dynamic and therefore on the heat and mass transfer as well. As a deep insight into the bed morphology is key to understand the fluid dynamic and all other related transport phenomena, several authors investigated the bed morphology of packed-bed reactors since the late 1950s.

3.1.1 Spherical particles

The majority of authors investigated the morphology of packed-beds filled with spherical particles. Beside nuclear applications, this particle shape is not very often used in the industry. But the understanding of this simple particle shape is the basis to comprehend the bed morphology of more complex shapes.

Augier et al. (2010) investigated a random packing of spherical particles using a soft-sphere DEM approach for a reactor with a diameter ratio of $N=25$ and a bed height of $17d_p$. The calculated radial void fraction profile is in good agreement with experimental measurements presented by Giese et al. (1998). The authors also analyzed the number of contact points that each particle shares with its neighbors. The majority of the particles are in touch with five particles next to each of them, resulting in a mean number of contacts per particle of $\bar{n}_c = 5.4$ and a standard deviation of $s = 1.5$.

Eppinger et al. (2011) generated packings with even numbered diameter ratios of $N=4 \dots 10$ using DEM. The radial void fraction distribution is in very good agreement with the ones predicted by using the correlation of de Klerk (2003). For $N=4$, the authors observed a significant increase of void fraction in the center of the reactor that cannot be described by using the correlation above but has also been found in experimental measurements by Krischke (2001). The high porosity in the center region indicates the formation of a channel along the center axis of the bed. This effect is known to become relevant for very low and even numbered tube-to-particle diameter ratios. In a later publication by Eppinger et al. (2014a), this was verified and validated with experimental data of Mueller (1992), as can be seen in Figure 15.

Most published data regarding the overall bed porosity and the radial void fraction profile of numerically generated packed-beds agree well with experimental data. Nevertheless, one needs to be careful as properties like the friction coefficient, particle flow rate, and effects like vibrations have significant impact on the bed voidage and the radial void fraction distribution. The bed gets more

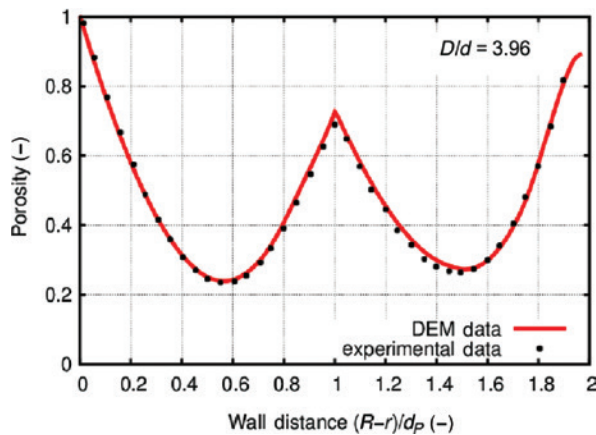


Figure 15: Comparison of the radial porosity profile for a monodisperse packed bed with $N=3.96$. Experimental data are taken from Mueller (1992). Reprinted with permission from Eppinger et al. (2014a).

dense with decreasing friction coefficients and increasing particle flow rate and due to vibrational effects.

3.1.2 Cylindrical particles

Cylindrical particles are often found in the chemical industry as catalyst support as they can be very easily manufactured using extruders. Compared to spherical particles, the bed morphology is determined not only by the position of the particles but also by their orientation.

Xu et al. (2006) were the first who numerically analyzed packings of cylindrical particles with a particle diameter of 5.4 mm, a particle length of 3.8 mm in a column with a diameter of 43.3 mm using DEM and a Monte Carlo method. They statistically analyzed the bed morphology and compared the results with experimental measurements using NMR. They found that DEM is able to generate packings that are comparable to experimentally generated beds in terms of integral indicators like packing density and mean empty space between solids in the packing. The usability of DEM for the generation of packings with cylindrical particles was confirmed and extended by Eppinger et al. (2014a). Regarding the global void fraction, they found very good agreement with correlations by Dixon (1988) and Foumeny and Roshani (1991). The radial void fraction distribution matched the experimental data of Giese et al. (1998) and Roshani (1990), as shown in Figure 16.

Caulkin et al. (2009b) and Boccardo et al. (2015) analyzed the orientation of the particles. Their findings indicate that the highest number of particles (around 20%) tends to be horizontally aligned with respect to their

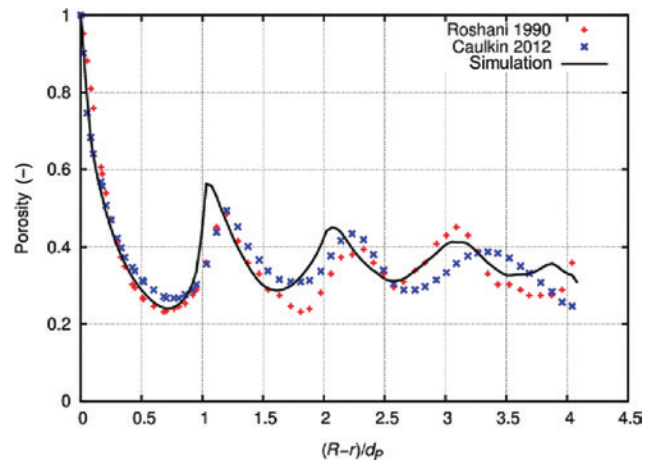


Figure 16: Comparison of the radial porosity distribution of a packed bed with cylindrical particles. Reprinted with permission from Eppinger et al. (2014a).

rotation axis. Furthermore, the results of Caulkin et al. (2009b) show that the axial void fraction profile flattens out into a constant value after a distance of four particle diameters from the bottom while the radial profile reaches a constant value about six particle diameters away from the wall. If the bed gets tapped after the filling process, the packing gets more dense. As a result, the oscillating radial void fraction profile gets more pronounced regarding its local minima and maxima and reaches a constant voidage after 10 particle diameters.

3.1.3 Complex particle shapes

Beside spherical and cylindrical particles, even more complex shapes are used in the industry, especially as catalyst support. The goal is to increase the active surface of catalyst particles and to achieve a more homogeneous plug-flow-like velocity field.

In several publications, Caulkin et al. (2008, 2009b, 2012) investigated the bed morphology of different complex particle shapes like hollow cylinders, multihole cylinders, Pall rings, and Trilobes.

Caulkin et al. (2009b) explored the impact of the inner diameter of hollow cylinders on the radial void fraction distribution and compared their results against a packing of four-hole cylinders. The results are depicted in Figure 17. They found that in case of hollow cylinders, the radial porosity profile strongly depends on the ratio of inner to outer particle diameter. For a small inner diameter, the radial void fraction distribution is comparable to that of cylindrical particles. Only the local minimum and maximum porosity values are more pronounced, and due

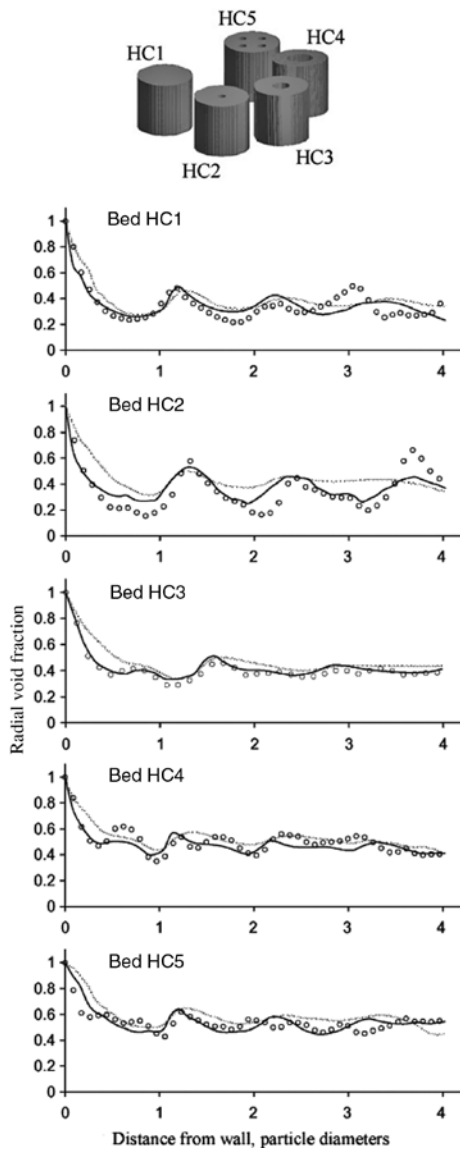


Figure 17: Investigated particle shapes and comparison of measured (symbol), DigiPac (dashed line), and DigiCGP (solid line) predicted local voidage data for conventional packed beds of hollow cylinders. Reprinted with permission from Caulkin et al. (2009b). Copyright (2009) Elsevier.

to the inner void of the particles, the local minima show some dents. If the inner diameter gets further increased, the small dents develop to pronounced additional local porosity maxima in the near-wall region (within $2d_p$ distance from the wall). Caulkin et al. (2008) found that those maxima get more pronounced if the bed gets tapped after the filling process. After a distance of approximately $2d_p$ away from the wall, an almost homogenous porosity distribution can be observed if the inner diameter chosen is not too small. Of course, the mean bed void fraction increases if the inner voidage of the particles is raised. The results

were confirmed by Eppinger et al. (2014b), who also found pronounced additional local porosity maxima within a distance of two particle layers away from the wall followed by a nearly homogeneous void fraction distribution.

The comparison of the radial void fraction profiles of hollow and multihole cylinders with equal inner voidage (HC3 and HC5) shows quite similar results. The multihole cylinders show less pronounced additional voidage maxima. This is due to the spreading of the inner void of a particle. In combination with the different orientation possibilities, this leads to an additional statistical smearing effect. Furthermore, the results indicate that multihole cylinders show an increased overall bed voidage. As the inner void volume is equal for both particle shapes, theoretically, the overall bed voidage should not change as long as the particle properties, the boundary conditions, or the particle dynamic during the filling process is not significantly changed.

Packings of Pall rings have been investigated by Caulkin et al. (2008). They found that packed beds of Pall rings show an unusual radial void fraction distribution within the near-wall region. Multiple local minima and maxima can be observed in that area. After a distance of about $3d_p$ away from the wall, the radial void fraction profile gets constant.

Caulkin et al. (2012) found that Trilobes ($h_p/d_p=4.6$) show a proximate constant radial void fraction distribution after a distance of $1.25d_p$ away from the wall. The porosity profile shows a local maximum of $\varepsilon=0.506$ at a distance of $0.82d_p$ followed by a global minimum of $\varepsilon=0.468$ at $0.5d_p$ followed by an increasing porosity towards the confining wall. It needs to be stated that the results for the near-wall region are at least questionable as the simulated void fraction and the experimental data as well do not reach a value of $\varepsilon=1$ directly at the wall.

3.2 Fluid dynamics

For fixed-bed reactors with a low N , the heterogeneous bed morphology strongly affects the fluid dynamics of the system. Local effects – especially in the near wall region – have significant impact on the characterization of the overall system. Therefore, common simplifications like a plug-flow-like radial velocity profile valid for extended packed-beds do not hold true for low diameter ratios N .

3.2.1 Pressure drop

One of the most important and crucial parameters for the design of a fixed-bed reactor is the pressure drop. It

determines the necessary energy for pumps and compressors to achieve a specified throughput. The specific pressure drop can be written in the form:

$$\frac{\Delta p}{L} = f_f \frac{\rho_f v_{in}^2}{d_p} \quad (73)$$

with L as the bed height, f_f as the friction factor, ρ_f as the fluid density, v_{in} as superficial velocity, and d_p as the particle diameter. Note that for a particle size distribution, the Sauter diameter d_{32} is often used instead of d_p . Several correlations exist to determine f_f . Probably the most famous one was developed by Ergun (1952).

The Ergun equation only takes into account frictional losses due to the packing and neglects wall effects. Furthermore, a homogeneous void fraction distribution is assumed. However, frictional losses from the wall of the tube have a significant effect on the pressure drop for low N . And as already discussed in subsection 3.1, the assumption of a homogeneous bed porosity does not hold true for small diameter ratios. Authors like Mehta and Hawley (1969) and Einfeld and Schnitzlein (2001) presented corrected friction factor correlations suitable for packed beds with low diameter ratios.

Wehinger (2016) compared different pressure drop correlations. In Figure 18, the ratio of friction factor from either Mehta and Hawley (1969) or Einfeld and Schnitzlein (2001) to the friction factor from the Ergun equation is plotted against Re_p for spherical and cylindrical particles. Different values for N are shown. The equation of Mehta and Hawley (1969) predicts friction factor ratios larger than unity over the entire range of Re_p . On the contrary, the correlation by Einfeld and Schnitzlein (2001) shows a transition from ratios above unity to

ratios below unity depending on Re_p and N . This is in agreement with observations made by Reichelt (1972), who found – compared to an extended packed bed – an increased pressure drop in the laminar and a decreased pressure drop in the turbulent flow regime. It can be seen that depending on particle shape, Re_p and N , the alternative correlations show significant deviations from the classical Ergun equation.

The developed pressure drop correlations for slim packed beds have the disadvantage that they rely on empirical parameters, which first need to be determined experimentally for different particle shapes. However, it is desirable to have a method that, in the early stage of the design process of a new particle shape, is able to predict the pressure drop accurately without relying on costly and time-consuming experimental measurements. Several authors have shown that CFD is such a tool.

Eppinger et al. (2011) and Wehinger et al. (2015a, 2017a), Wehinger (2016) have shown that they are able to match the data of Einfeld and Schnitzlein (2001) for spherical and cylindrical particles with an accuracy of $\pm 15\%$ (see Figure 19) for $Re_p = 10 \dots 1000$ by using polyhedral cells and the local caps meshing approach (see subsection 2.2 for further details). Ookawara et al. (2007) used the local bridges meshing strategy and a tetrahedral mesh and were also able to meet the results of Einfeld and Schnitzlein (2001) for spherical particles and $Re_p = 0.5 \dots 500$.

For an accurate prediction of the pressure drop, it is essential to use meshing strategies that use a local contact point treatment like the local caps or bridges method. The effect of global modifications on the bed porosity and, therefore, on the pressure drop is too significant to achieve precise results.

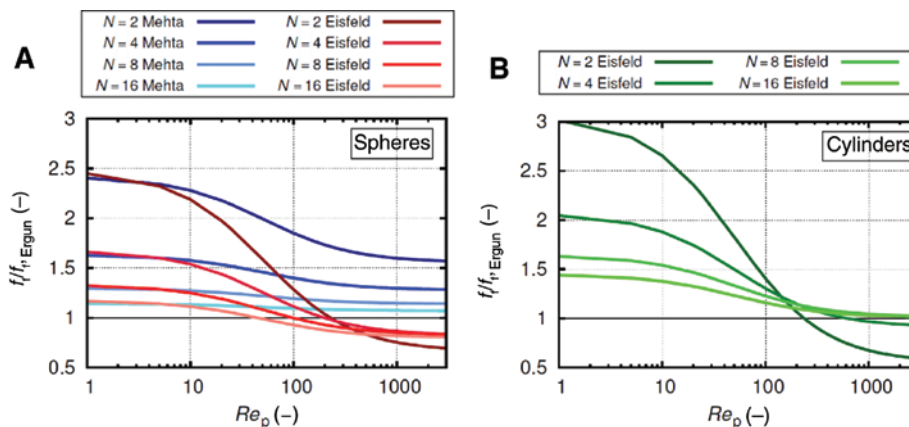


Figure 18: Ratio of friction factor from correlations and Ergun equation over Re_p for (A) spheres and (B) cylinders. Reprinted with permission from Wehinger (2016).

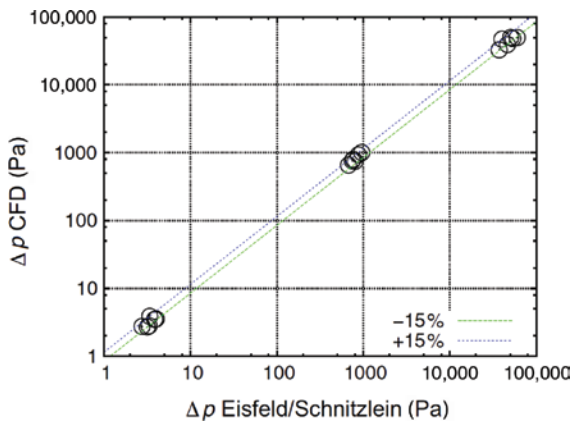


Figure 19: Pressure drop over the fixed-bed: the data cloud on the left-hand side represent different D/d_p ratios at $Re_p = 1$, the central one at $Re_p = 100$, and the right-hand side at $Re_p = 1000$. Reprinted with permission from Eppinger et al. (2011). Copyright (2011) Elsevier.

3.2.2 Velocity field

The complex interplay between bed morphology and local velocity field has been investigated by several authors

during the last decade. Although the velocity field shows strong local differences, it was found that the radial distribution of the axially averaged velocity field can be correlated to the radial void fraction profile.

Eppinger et al. (2011) investigated the flow field of packed beds made of spherical particles for several diameter ratios ($N=5\dots8$) and Reynolds numbers ($Re_p=1\dots1000$). They found that the maximum value of the axial velocity can exceed the superficial inlet velocity by a factor of 11. Those maximum velocities are found in the interstitial voids, while the areas in the vicinity of the contact points are characterized by stagnant zones of very low velocity. Furthermore, a strong wake can be seen at the end of the fixed bed.

Figure 20 shows the radial distribution of the void fraction and the axially and circumferentially averaged normalized axial velocity for different N and Re_p . A strong correlation between the velocity and void fraction profile can be found. Due to the wall effect, the highest velocity exists in the proximity of the wall. It exceeds the theoretical value of the mean interstitial velocity by a factor of three, almost independent of N and Re_p . With increasing distance from the wall, the velocity profiles show a declining oscillating behavior, whereas the local minima and

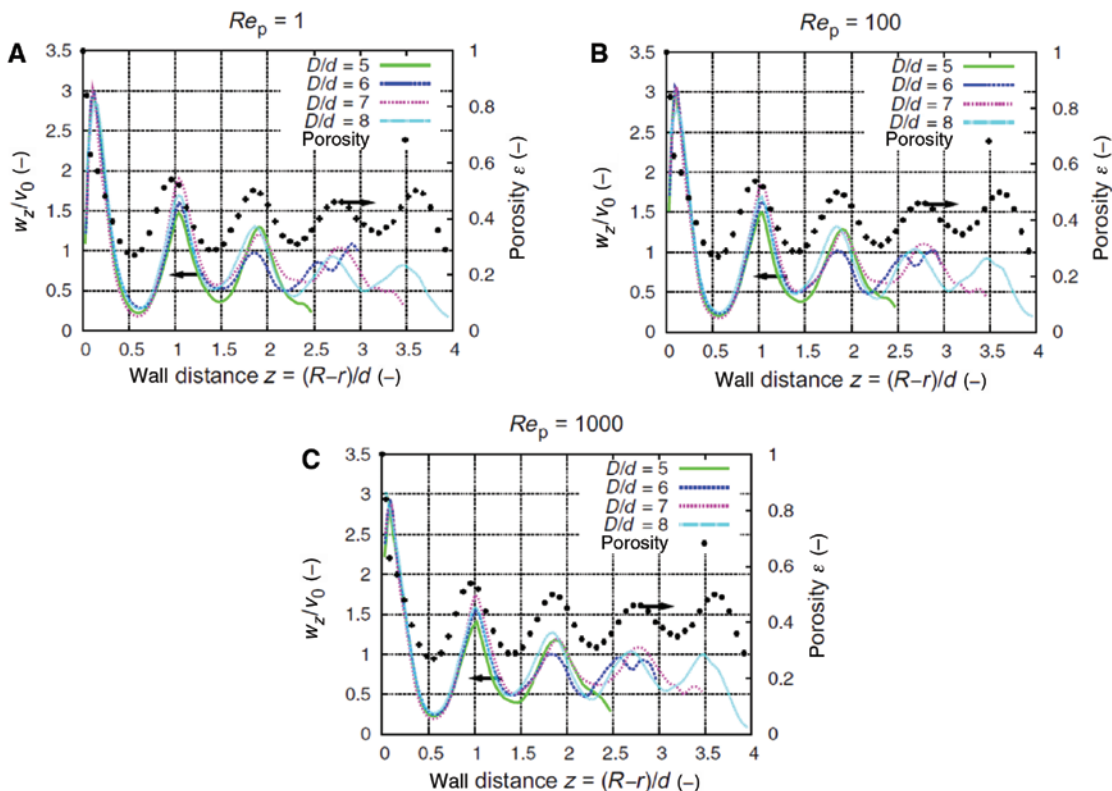


Figure 20: Circumferential-averaged axial velocity profile calculated from velocity values within the bed for (A) $Re_p = 1$, (B) $Re_p = 100$, and (C) $Re_p = 1000$ and different diameter ratios.

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maxima correspond with the oscillations of the radial void fraction distribution.

The impact of the particle shape on the velocity field was investigated by Wehinger (2016). He compared spheres, cylinders, and one-hole cylinders and found that spherical and cylindrical particles show a quite similar radial profile of the averaged velocity, as can be seen in Figure 21. The numerical results agree well with the experimental data of Giese et al. (1998) and show a maximum velocity of approximately two times the local axially and circumferentially averaged interstitial velocity close to the wall. For one-hole cylinders, the bypassing effect near the wall is significantly reduced by showing a maximum velocity of 1.7 times the local interstitial velocity. This is caused by additional inner particle voids that do not only reduce the near wall bypassing effect but also lead to an overall more homogeneous velocity field that shows a less oscillating behavior. This is in accordance with simulation results of Caulkin et al. (2008, 2012) and data of Giese et al. (1998) and Bey and Eigenberger (1997). For more complex particle shapes like Trilobes, Caulkin et al. (2012) found an almost plug-flow-like velocity profile.

The impact of the inner particle voids on the local flow field was examined by Dixon et al. (2012a). Based on a segment model, they compared the flow field characteristic of cylindrical particles containing a different number of holes. They found that additional holes reduce regions of stagnant and swirling flow behind the particle. Whether the holes are effectively used depends on hole diameter and particle orientation. The more the particles are oriented perpendicular to the flow, the less effective are the inner voids. This indicates the importance of knowledge of the particle orientation as additional parameter to describe the bed morphology.

3.2.3 Mixing characteristics

The problem with axial and circumferential averaging the velocity field is that all information regarding possible backflow and stagnant zones is lost. But the knowledge about these effects is essential, as it can affect safety and efficiency in a negative manner.

Eppinger et al. (2011) investigated stagnant zones and backflow regions for packings of spherical particles, as shown in Figure 22. They found that the frequency of these phenomena increases for higher Reynolds numbers. For $Re_p = 1000$, around 13% of the packed bed is characterized by either stagnant zones or backflow, while in the laminar flow regime, only 1% of the volume is affected.

To quantify the effect of back mixing and stagnant zones, Wehinger (2016) investigated the residence time distribution of packed beds by conducting virtual tracer experiments using a particle tracking method. Figure 23 shows the residence time distribution functions for spheres, cylinders, and one-hole cylinders as a function of residence time normalized by the hydraulic residence time $\bar{t} = L/u_e$. The modal value of a distribution is the value that appears most often. All of the governed normalized modes show a residence time that is significantly lower than one, which indicates channeling effects. Furthermore, multiple decaying peaks occur, which represent strong internal recirculations and channeling. All curves show long tailing, which is a strong indicator for stagnant backwaters. The large variances reflect the large deviation from plug flow behavior. The one-hole cylinder packing shows the most plug-flow-like characteristic compared to the other particle shapes. It has the largest modal value and the narrowest distribution function with the least tailing.

Based on the radial distribution function, the axial dispersion coefficient D_{ax} can be determined, as done by

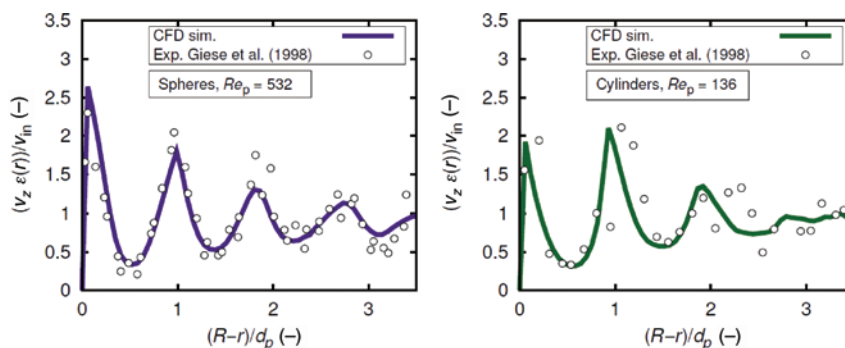


Figure 21: Specific velocity $(v_z \cdot \varepsilon(r)) / (v_{in})$ for experiments from Giese et al. (1998) and CFD simulations for packed bed of spheres (left) and cylinders (right).

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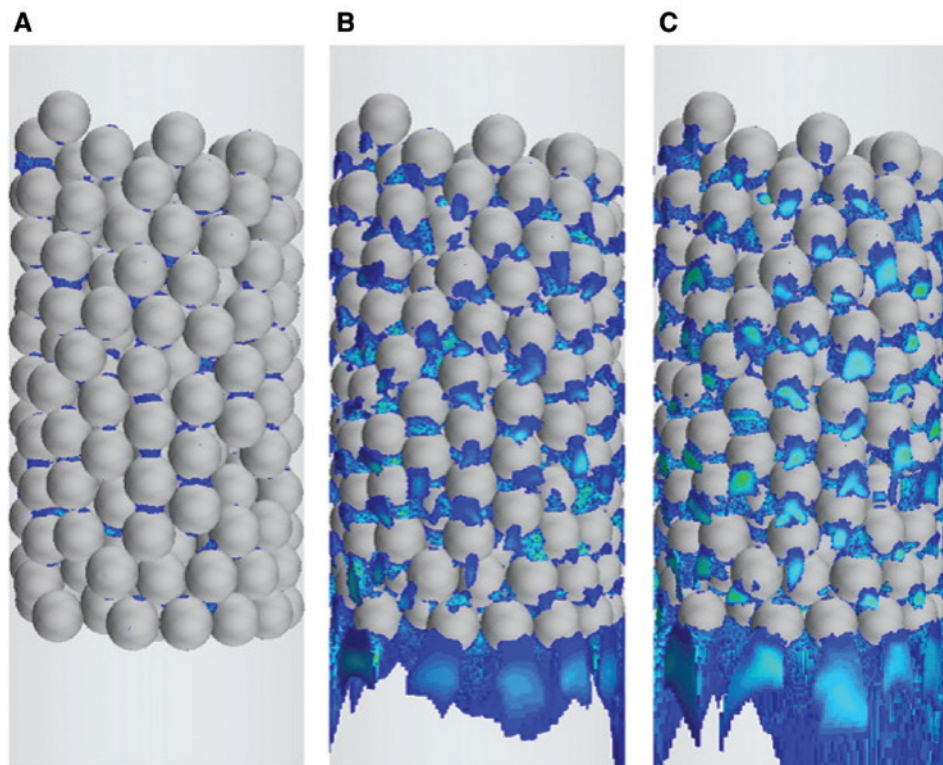


Figure 22: Regions with zero or negative velocities for (A) $Re_p = 1$, (B) $Re_p = 100$, and (C) $Re_p = 1000$. Reprinted with permission from Eppinger et al. (2011). Copyright (2011) Elsevier.

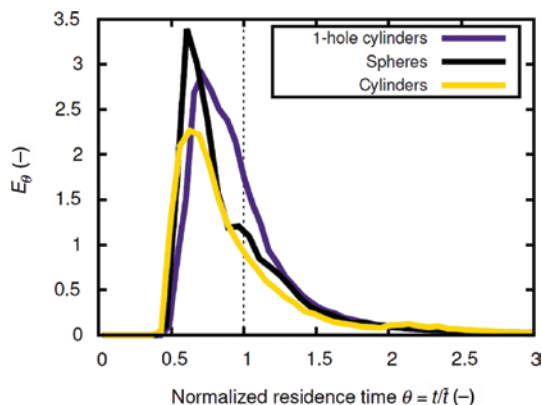


Figure 23: Residence time distribution function for different packings. Reprinted with permission from Wehinger (2016).

Freund et al. (2003, 2005) and Atmakidis and Kenig (2014). The dispersion coefficient, respectively, the Péclet number $Pe = (v_{in} \cdot d_p) / D_{ax}$, is not only a useful parameter to qualitatively characterize the system within the range of the plug flow reactor ($Pe \rightarrow \infty$) and the ideal stirring vessel ($Pe \rightarrow 0$) but also needed as input parameter for less sophisticated but faster modeling approaches like the axial dispersion model, which is widely used in the industry to calculate

the species propagation in fixed beds. Unfortunately, correlations for the axial dispersion coefficient are most often only known for simple particle shapes in extended fixed beds. Likely, it is a useful option to determine the axial dispersion coefficient based on particle-resolved CFD simulations, which can later be used in simplified models. By now, published CFD studies dedicated to dispersion in fixed-bed reactors are small in number and only limited to $Re_p < 300$, which is below most industrial applications.

3.3 Heat transfer

An adequate description of both the reaction kinetics and the transport processes of heat and mass is required to design a (catalytic) fixed-bed reactor. In Figure 24, different mechanisms for heat transfer are illustrated schematically in a fixed bed of spheres. Depending on process parameters like flow rate, temperature, and fluid properties, certain mechanisms can dominate.

The radial transport process of heat is of paramount interest, as it is the direction of heating or cooling the reactor. Especially, for low N fixed beds, an accurate description of the radial transport is difficult. Dixon (2012)

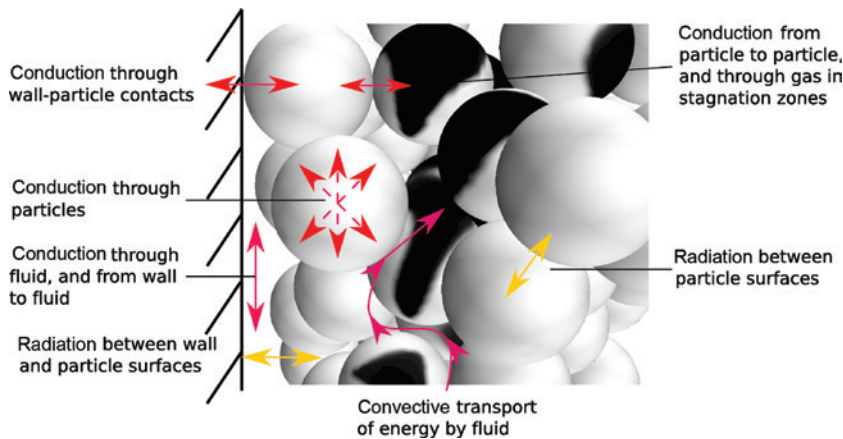


Figure 24: Different mechanisms for heat transfer in a fixed-bed reactor. Reprinted with permission from Wehinger (2016).

comprehensively reviewed radial heat-transfer models in fixed-bed catalytic reactors. The most common formulation is the $k_r - h_w$ model, which is based on the classical pseudo-homogeneous 2D (radial and axial direction) axially dispersed plug flow (ADPF) model. In Europe, this model is called “ α_w ” model. The boundary condition at the wall ($r = R$) yields:

$$-k_r \frac{\partial T}{\partial r} = h_w (T - T_w), \quad (74)$$

where k_r is the constant effective radial thermal conductivity, T is temperature, T_w is wall temperature, r is the radial coordinate, and h_w represents the wall heat transfer coefficient. This model shows a temperature “jump” at the wall, due to h_w Dixon (2012). In most cases, h_w is expressed by the wall Nusselt number Nu_w :

$$Nu_w = h_w d_p / k_f, \quad (75)$$

where k_f is the thermal conductivity of the fluid.

Several formulations for Nu_w are available in literature. However, the most recommended form is the mechanistic model. Nu_w represents the sum of the contribution of the decreased solid-phase conduction $Nu_{w,0}$ and the contribution of the decreased lateral convective heat transfer near the wall $a \cdot Pr^b Re^c$. Martin and Nilles (1993) formulated Nu_w as follows:

$$Nu_w = Nu_{w,0} + 0.19 Pr^{1/3} Re_p^{3/4}, \quad (76)$$

where the Prandtl number is defined as $Pr = c_p \mu / k$. Dixon (2012) recommends to calculate $Nu_{w,0}$ with:

$$Nu_{w,0} = \left(1.3 + \frac{5}{N} \right) \left(\frac{k_r^0}{k_f} \right), \quad (77)$$

with $N = d_p / D$, and k_r^0 is the thermal conductivity of the bed, which can be calculated by the Zehner-Schlünder equation, see, e.g. Dixon et al. (2013a). In literature, there are more formulations found especially for the decreased lateral convective heat transfer near the wall. A critical review on different models and recommendations was given by Dixon (2012). The authors conclude that the $k_r - h_w$ model shows likely an error range of 20–30%. Under these circumstances, particle-resolved CFD simulations of packed beds can shed light on the radial heat transfer, especially for low N beds. Magnico (2009) performed transient DNS of heat transfer in a bed with 326 and 620 particles, respectively. The author found reasonable agreement for h_w with the correlation of Martin and Nilles (1993). Full-bed CFD simulations with an accompanying experimental measurement are the exception. Dixon et al. (2012b) simulated a full bed of 1000 and 1250 spheres including heat transfer through the particles. Radial temperatures were measured at the outlet of the packed bed. The agreement between experiment and simulations is reasonable. The authors tried to examine steeper gradients inside the bed by varying the bed length.

More recently, axial heat transfer by particle-resolved 3D CFD simulations was validated with axial experimental data by Wehinger et al. (2016b). For the experiments, a reactor setup was used, see Figure 25, which measures the axial temperature and, if present, the axial concentration *in situ* and *operando* with a capillary technique. Through a small orifice, a small amount of gas is constantly sucked into the capillary giving the axial gas-phase temperature. Two sets of experiments were conducted without chemical reactions, see Figure 26, and compared to simulated values. In the detailed simulations, the capillary was

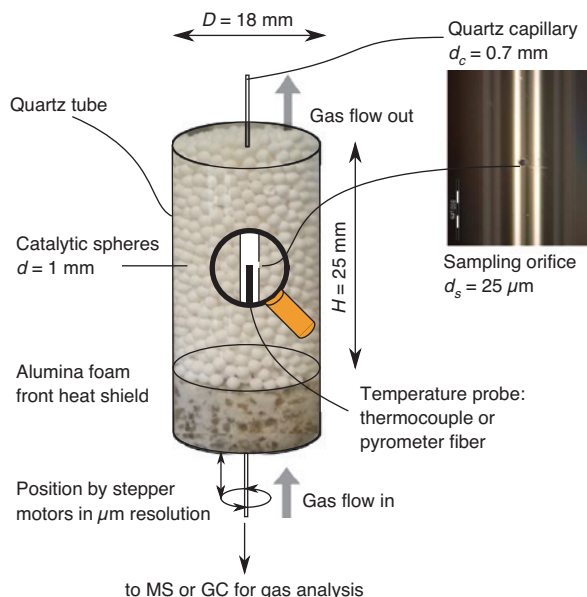


Figure 25: Profile reactor set-up with spherical particles. Reprinted with permission from Wehinger et al. (2016b). Copyright (2016) John Wiley and Sons.

taken into account around which the spheres settled. For the simulations, the values of the adiabatic capillary wall were used, representing the axial corresponding gas-phase temperature. As can be seen, the CFD simulations show an excellent agreement with the experiments. This case highlights that the local transport of heat inside a fixed-bed reactor can be reproduced with high accuracy over the entire length by the particle-resolved CFD approach.

Dong et al. (2017) investigated radial heat transfer by profile measurement techniques and 3D CFD simulations. Heat transfer in a small D fixed-bed reactor packed with spheres and one-hole cylinders was studied at different flow rates ($60 < Re_p < 100$) and packing heights. In general, good agreement between experiment and simulation was achieved. However, in the simulations, the influence of the capillary inserted in the packed bed was not taken into account. In future studies, the quantitative effect of the capillary technique on the observed heat and mass transfer should be studied. Especially, industrial relevant reactor conditions ($Re_p > 1000$ and temperatures up to 1300 K) are of interest.

As pointed out in subsection 2.2, particle-particle and particle-wall contacts can lead to severe convergence problems. Hence, a modification of these contacts is required for most of the CFD simulations. Recently, Wehinger et al. (2017a) studied the effects of local contact modifications (i.e. caps and bridges method for line and area contacts,

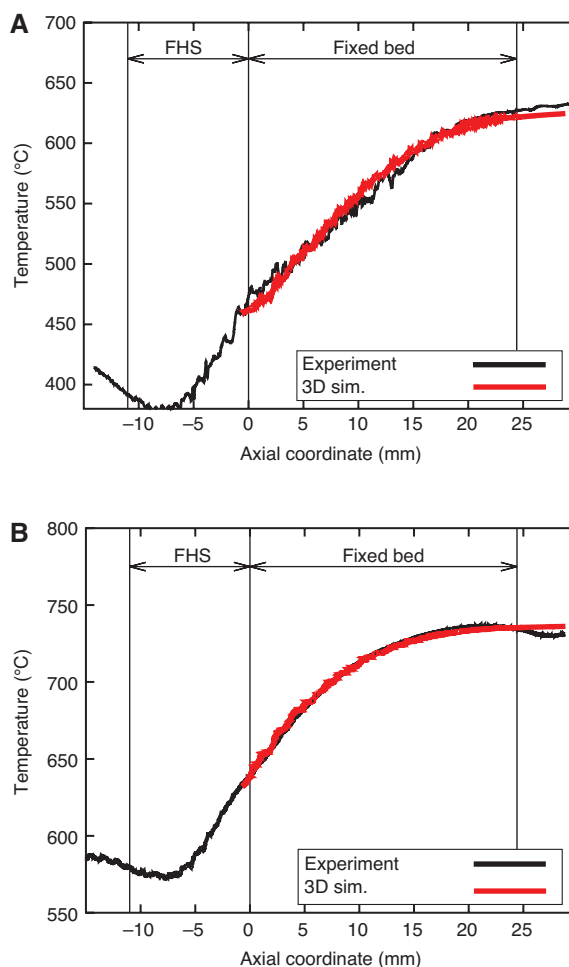


Figure 26: Measured and simulated axial temperature profile. Front heat shield (FHS). (A) $T_{\text{wall}} = 635^\circ\text{C}$ and $\dot{V}_{\text{in}} = 2500$ ml/min. (B) $T_{\text{wall}} = 735^\circ\text{C}$ and $\dot{V}_{\text{in}} = 1500$ ml/min. Reprinted with permission from Wehinger et al. (2016b). Copyright (2016) John Wiley and Sons.

and caps and united method for overlaps resulting from composite DEM particles) towards heat transfer in a bed of cylinders. Besides heat transfer, pressure drop and local velocity are investigated. The study shows that with increasing Re_p , the modification of particle-wall contacts is becoming more important, see Figure 27. As convective heat transfer is becoming dominant, contact modifications in the bulk of the bed are becoming less important. Promising results in terms of heat-transfer predictions can be achieved with the local caps method. However, for high flow rates, convective heat transfer in contact regions is overestimated. The alternative bridges method predicts well pressure drop and heat transfer. Nonetheless, the choice of the thermal conductivity of these artificially inserted bridges is still a topic to discuss much more thoroughly.

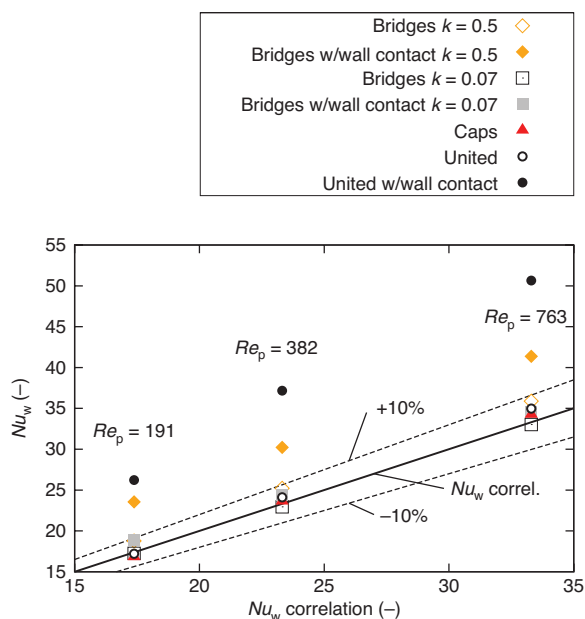


Figure 27: Parity plot of wall Nusselt number from CFD simulations of different contact-area modifications over Nu_w from correlation of Martin and Nilles (1993).

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3.4 Mass transfer and heterogeneous reacting systems

Dixon et al. (2006) reviewed comprehensively the contributions of particle-resolved CFD simulations incorporating mass transfer and reacting systems until 2006. Only work after that date is reviewed here. In the section, we particularly focus on reactor geometry, type of mesh, flow conditions, applied kinetics, and coupled transport phenomena.

3.4.1 Mass transfer and homogeneous reactions

Guardo et al. (2007) investigated particle-to-fluid heat and mass transfer in supercritical conditions by means of particle-resolved CFD simulations. This work is a continuation of earlier contributions where heat transfer and the influence of turbulence were studied with detailed CFD in a randomly packed bed (Guardo et al. 2004, 2005). The authors used a bed consisting of 44 spheres with $N=3.923$, which is the same arrangement as in the previous studies. Increasing the spheres by 0.5% avoided contact point problems, i.e. the global overlaps method. The CFD code Fluent was used to construct the tetrahedral mesh and solve the Navier-Stokes equations including the

Spalart-Allmaras turbulence model. Constant temperature and C_7H_8 concentration on the surface were taken into account to evaluate transport coefficients for heat and mass (k_c, h). The CFD results were in good agreement with correlations from literature for $Re_p > 10$. However, a strong dependency on the mesh was noticed.

Mousazadeh et al. (2013) presented a 2D model of a randomly packed fixed-bed reactor of spheres with a length of $L=60 \cdot d_p$ and a width of $D=10 \cdot d_p$. The arrangement of the spheres were conducted manually. An unstructured triangular mesh in the gas-phase bulk and 15 layers of quadrilateral cells were used. Three different meshes were tested with a total number of cells ranging from 1.8×10^6 to 2.7×10^6 . Intraparticle heat and mass transport were not considered. Laminar flow with Reynolds number of 3.5 based on particle diameter and interstitial velocity was studied using the CFD software Fluent. The gas-phase reaction between ethylene and oxygen to ethylene oxide was considered by using a one-step Arrhenius type kinetic. The axial development of species concentration and temperature were in agreement with a plug flow model.

3.4.2 Heterogeneous catalytic reactors

Fixed-bed reactors are commonly used for gas-solid catalyzed reactions, such as ammonia synthesis, syngas production (see Figure 28), hydro-cracking, etc. (Eigenberger 2008). As a consequence, the inclusion of surface reactions into the modeling of fixed-bed reactors is highly relevant

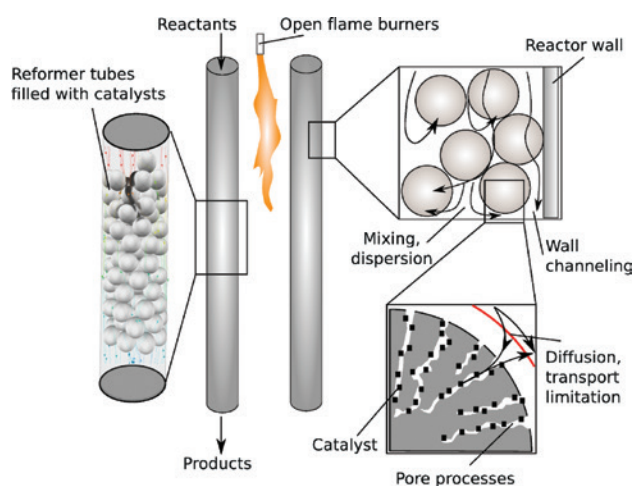


Figure 28: Typical arrangement of reformer tubes filled with catalytic particles.

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for industrial applications. Industrial catalysts have to meet economic requirements, i.e. space-time yield, activity, selectivity, and costs. In general, three different structural types of solid catalysts are applied, which depend on the chemical reactions to be catalyzed (Gallei et al. 2008, Deutschmann et al. 2009):

1. Unsupported (bulk) catalysts consists completely of the catalyst material. Examples are metal oxides, simple binary oxides, complex multicomponent oxides, carbons, etc.
2. Supported catalysts consists of support on which the catalyst material is dispersed. The support, which can be also catalytic active, is sponge-like and characterized by a high porosity and large internal surface. Most commonly binary oxides, like transitional alumina, $\alpha\text{Al}_2\text{O}_3$, SiO_2 , etc., are applied. Typical arrangements are shown in Figure 29.
3. Coated catalysts are indicated by a thin layer ($\approx 100\ \mu\text{m}$) or shell of active material allocated across an inert structured surface. Examples are egg-shell catalyst, monolithic honeycombs, structure packings, foams, catalytic filters, etc.

This section is divided into CFD models that only take into account reactions on the pellet surface and such models that include reactions inside the pellets, i.e. intraparticle reactions.

3.4.2.1 Surface chemistry only

Models that only take into account surface chemistry while intraparticle mass transport is neglected are oriented towards egg-shell catalyst configurations, see Figure 29B.

A study of endothermic SRM by Kuroki et al. (2009) considered 349 spheres of 1.75-mm diameter in an annular bed of 25-mm length with inner diameter of 2 mm and outer diameter of 10 mm. DEM simulations were used to generate random packing. The so-called bridge method was utilized to prevent convergence problems close to contact points. Finally, a tetrahedral volume mesh was

created with refinement close to surfaces, which was developed in an earlier study (Kuroki et al. 2007). Intraparticle heat and mass transport was not considered. On the catalyst surface, a three-step reaction mechanism of SRM developed by Xu and Froment (1989) was implemented. The rate constants were re-defined to meet the definition of a molar flux through the particle surface, i.e. in units of $\text{mol m}^{-2} \text{s}^{-1}$. Subroutines evaluating the reaction rate were included into the CFD software Fluent. The authors studied SRM with varying wall heat flux and Re_p number of 300. The outlet species concentrations were compared to experimental data from Hoang et al. (2005) and agreed remarkably well, especially when methane was nearly consumed. However, transport properties inside the bed were not validated.

Li et al. (2013) modeled the endothermic dehydrogenation of isopropanol in a 2D Sierpinski carpet fractal structure representing a porous medium of the fixed bed. A Lattice-Boltzmann code was used in which a one-step Arrhenius type kinetic was implemented. Intraparticle phenomena were not incorporated. Laminar flow at 450 K was studied without any comparison to experimental data.

A catalytic fixed bed of spheres with varying diameters ($N=2.3, 3.2, 5.3$) and height of $H/d_p=7.2, 10$, and 16.6 was modeled and spatially resolved by Zhou et al. (2013). Acetone hydrogenation was considered on the adiabatic surfaces with a one-step LHHW reaction-kinetic model. A non-specified structural mesh was used in Fluent to calculate this problem. The effect of different velocities at laminar flow towards reactor performance and pressure drop was tested. The authors concluded that smaller diameter pellets are advantageous for acetone hydrogenation, but large pressure drop is the downside.

Particle-resolved CFD was used in a couple of studies to investigate details of coal combustors and gasifiers (Safronov et al. 2014, Richter et al. 2015, Sahu et al. 2016). The studies ranged from reacting single spheres to 2D arrays of spheres and finally to a porous particle made up of an agglomeration of 185 monodisperse spherical particles. Semi-global heterogeneous and homogeneous reactions following Arrhenius formulations were applied. Laminar flow, ambient pressure, surface radiation, and temperatures up to 3000 K were studied. Intraparticle heat and mass transport were not taken into account. ANSYS Fluent was used to generate the prismatic or unstructured mesh ranging from several thousands cells in case of the 2D domain to several million cells for the porous particle case. Interactions of local kinetic and local transport phenomena were evaluated using the Thiele modulus, effectiveness factor, and Damköhler numbers.

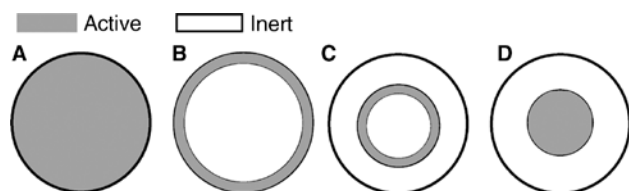


Figure 29: Typical catalyst distribution inside a support. (A) Uniform, (B) egg shell, (C) egg white and (D) egg yolk. Adapted with permission from Lekhal et al. (2001). Copyright (2001) Elsevier.

Microkinetics describe heterogeneous catalysis in a very fundamental way. Maestri and Cuoci (2013b) coupled full 3D CFD with microkinetics description of H_2 fuel-rich combustion on Rh. A novel solver, called catalyticFOAM (Maestri and Cuoci 2013a), was proposed being based on the operating-splitter technique and incorporated into the software environment OpenFOAM. As a show case, the new solver is tested on two non-touching adiabatic spheres with a diameter of 3 mm in a tube of 5 mm in diameter. A 2D axisymmetric mesh was used consisting of 4000 cells of rectangular and triangular shapes. Full consumption of oxygen was reached in the wake of the spheres leading also to the adiabatic temperature of the reacting mixture. By using microkinetics, the most abundant reaction intermediate (MARI) can be specified, which is H^* in this case. The authors also presented an isothermal hydrogen fuel-rich combustion on Rh in a one-hole cylinder ring arrangement. The 3D model consisted of a unstructured hexahedral mesh of approximately 250,000 cells. Laminar flow at 473 K was considered. Surface reactivity and concentration profiles showed strong gradients, mainly due to the random arrangement of the bed. Again the microkinetic model could detect adsorbed species on the pellet surface. H^* was the MARI, which covered approximately 80% of the entire pellet surface, see Figure 30. As can be seen in this figure, the applied meshing algorithm was not able to produce a smooth surface of the pellets. Thin and sharp cells are apparent at the edges of the one-hole cylinders. Still, this work represents the initial distribution of coupling microkinetics of surface chemistry with detailed fixed-bed CFD.

Wehinger et al. (2015a) used the software STAR-CCM+ to couple the microkinetics description of dry reforming of methane (DRM) on Rh with full 3D in a bed of 113 spheres. The bed measured $H/d_p = 10$ and $N = 3.96$. The random bed generation was realized with DEM simulations. Locally

flattening of particles, i.e. the caps method, was applied to avoid bad cells. The locally refined polyhedral mesh consisted of approximately 3–11 million cells depending on the inlet velocity. Two prism layers close to the catalytic wall were generated to resolve appearing gradients. The prism layer thickness was approximated based on the boundary layer thickness in the stagnation point of a sphere (Dhole et al. 2006). Figure 31 shows the gas-phase mesh only with prism layers close to surfaces and polyhedral cells elsewhere. Intraparticle heat transfer was considered. The microkinetics description of the DRM was implemented as a species boundary condition on the surface of the spheres. Laminar and turbulent flow at constant inlet concentrations was studied resulting in different radial and axial properties. Although the computational time was very high, with such detailed CFD simulations, the interactions between local kinetics and local transport phenomena could be quantified, supporting fundamental understanding of fixed-bed reactors. This modeling approach was later extended towards fixed beds of spheres, cylinders, and one-hole cylinders (Wehinger et al. 2015b). The effect of pellet shapes towards the performance of a fixed-bed reactor was demonstrated with particle-resolved CFD for DRM on Rh. Besides evaluating axial, radial, and exit transport properties, the interpretation was assisted by residence-time distribution analysis. As can be seen in Figure 32, the local temperature influences the local mole fractions and vice versa. Figure 33 shows local variations of surface adsorbed carbon (C^*) for different pellet shapes of a DRM showcase. For this particular parameter set, the bed of cylinders showed the best performance regarding conversion and yield.

In a more recent study, Wehinger et al. (2016a) validated particle-resolved CFD simulations of DRM on nickel in a bed of spheres with experimental data. The

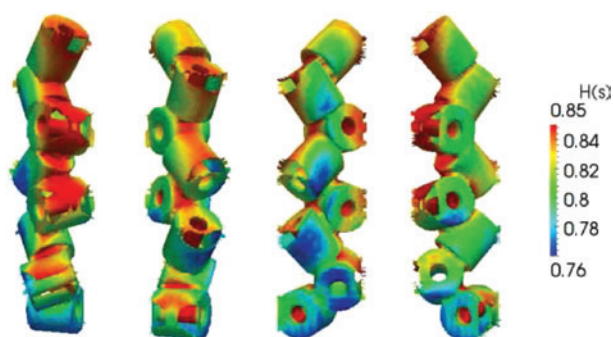


Figure 30: Surface adsorbed hydrogen (H^*) on an isothermal bed of one-hole cylinders. Reprinted with permission from Maestri and Cuoci (2013b). Copyright (2013) Elsevier.

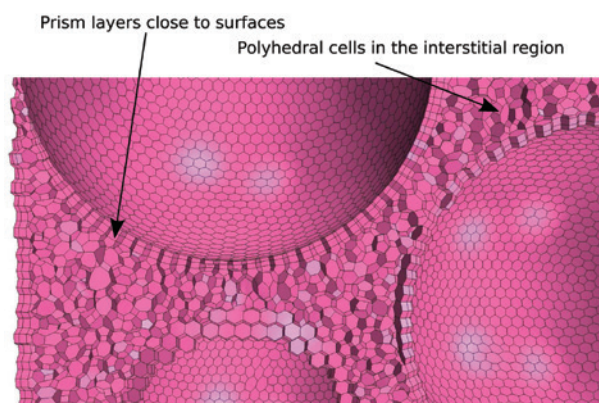


Figure 31: Mesh detail of CFD simulation. With permission from Wehinger (2016).

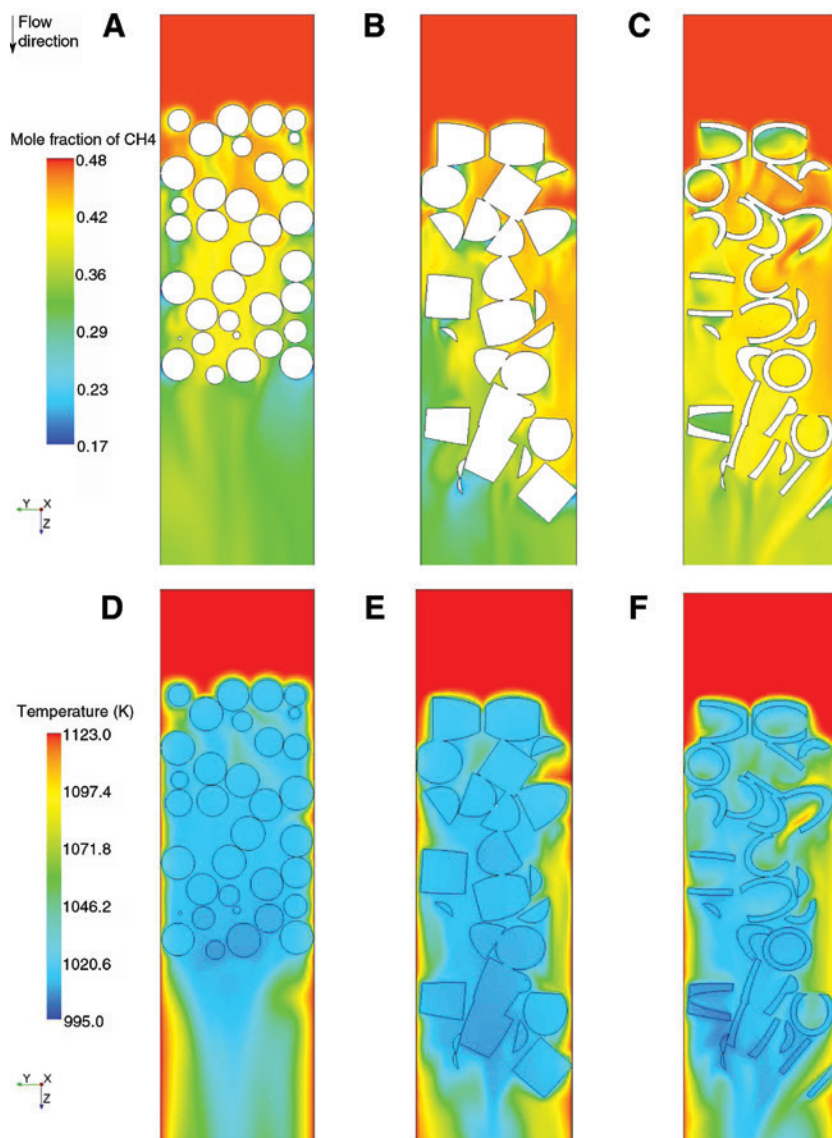


Figure 32: Temperature and mole fraction of methane are shown on a cut through a bed of spheres, cylinder and one-hole cylinders. Mole fraction of methane (A–C) and temperature (D–F) on a plane cut through the fixed-bed for spheres, cylinders and one-hole cylinders. Reprinted with permission from Wehinger et al. (2015b). Copyright (2015) John Wiley and Sons.

1-mm-diameter spheres had a thin washcoat ($<10\ \mu\text{m}$) in which the Ni catalyst was dispersed. In the profile reactor of the Horn group from TU Hamburg (Horn et al. 2010), axial temperature and species concentration profiles were measured. The bed was $H/d_p = 25$ high and $N = 18$ in diameter. With DEM simulations, the randomly packed bed of spheres including the measure capillary was generated. A comprehensive microkinetics from literature (Delgado et al. 2015) consisting of 52 irreversible reactions was used. Only a small slice of the entire bed was simulated due to the large amount of spheres in the bed. The mesh generation was conducted by the suggestions of the previous studies. Whereas heat transfer simulations without

reaction were in excellent agreement with the experiments, DRM could not be reproduced entirely. The main reason can be found in thermodynamic inconsistencies of the microkinetics for this set of experimental conditions. Inconsistencies in enthalpy led to a stronger decrease in the center of the bed, which had a great influence to the reactivity. Internal and external mass transport limitations were evaluated by using the Thiele modulus and the Damköhler number, respectively. As intraparticle limitations were moderate to small, the assumption of instantaneous diffusion was justified. The profile reactor setup from the Horn group delivers axial data with which particle-resolved CFD simulations can be validated in a very

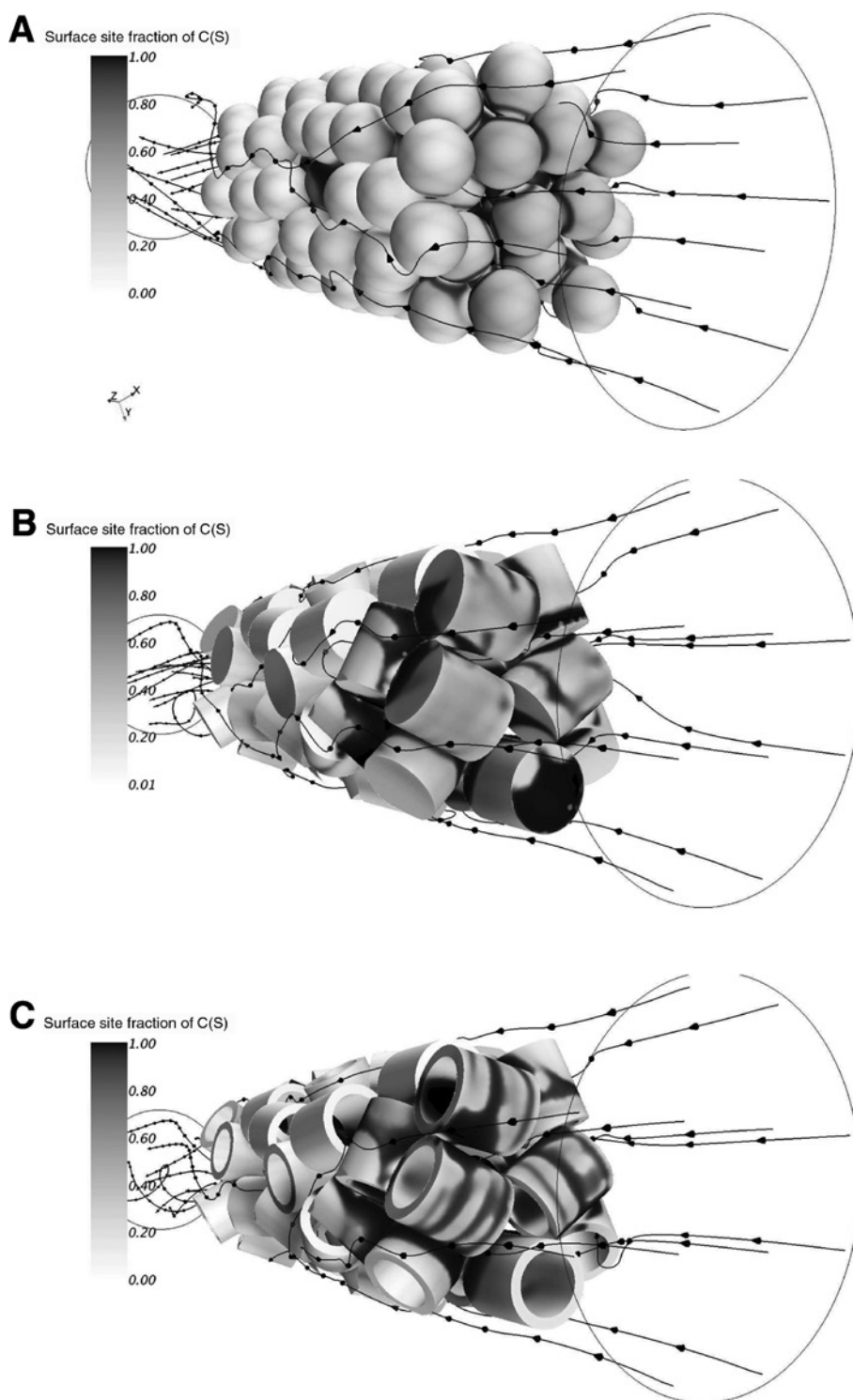


Figure 33: Surface site fraction of adsorbed carbon C^* and streamlines for (A) spheres, (B) cylinders, and (C) one-hole cylinders for the DRM. Reprinted with permission from Wehinger et al. (2015b). Copyright (2015) John Wiley and Sons.

precise manner. In the future, this excellent test reactor should be complemented with a radial capillary to detect the very important radial gradients in temperature and species concentration.

Another example of coupling microkinetics with particle-resolved CFD simulations was presented by Bracconi et al. (2017). In a duct containing 25 spheres, the catalytic methane conversion to syngas on Rh was modeled. For the

generation of the random packing, the LIGGGHTS open source CFD-DEM code (Kloss et al. 2012) was used. The bridges method was utilized avoiding contact points. The hexahedral mesh was generated with snappyHexMesh by OpenFOAM accounting for approximately 460,000 cells. The transient simulations were conducted under isothermal conditions and by assuming that the reaction occurs only at the surface of the catalytic spheres. The ISAT algorithm was applied for the calculation of the species source terms. The intention of this study was to demonstrate the reduction of calculation time, which was a factor 14 times faster than the direct integration of each time step.

3.4.2.2 Intraparticle mass transport and reactions

Models that account for intraparticle mass transport and reaction are oriented towards unsupported (bulk) catalyst configurations, uniformly supported catalysts, see Figure 29A, as well as the “egg white” and “egg yolk” configuration, i.e. Figure 29C and D. A common issue in modeling intraparticle mass transport and reactions is associated with the particle-fluid interface. The common porous media model used for particles has the serious deficiency that convective fluxes across that interface are present. This means that the no-slip condition at the particle surface is violated. Hence, this method is not suitable to model intraparticle mass transport and reactions in fixed-bed reactors. Dixon et al. (2010) presented a novel approach that models the particles as a solid with no-slip condition at the surface. User-defined scalars are formulated in the fluid and solid region, whereas reactions are defined in the solid region. Under steady-state conditions, the transport of scalar ϕ_k can be written as follows (Dixon et al. 2010):

$$\nabla \cdot (\rho \bar{\mathbf{V}} \phi_k - \Gamma_k \nabla \phi_k) = 0, \quad (78)$$

where $k=1, 2, \dots, N_{sp}-1$ is the number of species, ρ is the density, $\bar{\mathbf{V}}$ is the velocity, and Γ_k is the transport parameter. In the solid phase and under steady-state conditions, it reads:

$$-\nabla \cdot (\Gamma_k \nabla \phi_k) = S_{\phi_k} \quad (79)$$

The equation accounts only for transport by diffusion and a source term S_{ϕ_k} generating species ϕ_k . In both regions, the equations are accompanied by the following:

$$\phi_{N_{sp}} = 1 - \sum_{k=1}^{N_{sp}-1} \phi_k \quad (80)$$

Furthermore, the method accounts for separate diffusion, i.e. effective Fickian diffusivity inside the solid

particle and molecular and turbulent diffusivity in the fluid region. The method was tested against the porous medium model for a single sphere and further applied in a wall segment of spheres for SRM using a LHHW kinetics. Turbulence was accounted for using the SST $k-\omega$ model. The authors used Fluent for the mesh generation consisting of orthogonal prism cells close to the surfaces and hexahedral cells elsewhere. A total of approximately 1.8 million cells were used for the wall segment study. The spheres were shrunk by 0.5% to avoid bad cells in the contact points. Fluent was also used to solve the governing equations. Figure 34 shows a comparison between the porous media model and the solid particle model highlighting the effect of the no-slip condition.

This method was further applied in a series of studies by the Dixon group. Methane steam reforming and propane dehydrogenation were studied in a segment of cylindrical pellets by Taskin et al. (2010). Detailed pellet surface and intraparticle temperature, species, and reaction rate distributions were evaluated for a near-wall particle. Non-uniform and non-symmetric variations were detected for surface and intraparticle transport properties. Behnam et al. (2010) studied carbon deposition in propane dehydrogenation in a segment of cylinders. Later, SRM was modeled to predict temperature and species profiles under the experimental SRM reaction conditions (Behnam et al. 2012). Figure 35 shows CFD of temperature and different mass fraction results for a stack of non-spherical pellets of SRM. Dixon et al. (2012a) investigated reaction rates, conduction, and diffusion inside catalyst particles with one-, three-, four-, and six-holes for SRM. In addition, the external flow and temperature fields near the heated tube wall were evaluated. The work especially highlighted the effects of the different pellet features towards overall performance. However, experimental validation data were not considered.

More recent studies of the Dixon group investigated different techniques to reduce the calculation time and likewise retain the accuracy of the kinetic model for particle-resolved CFD simulations (Partopour and Dixon 2016a,b, 2017). Intraparticle transport and reaction were modeled with the solid particle method (Dixon et al. 2010). An illustrative example of temperature and species distribution in the gas phase and solid particle is shown in Figure 36 for a small bed of spheres in which ethylene oxidizes. In one study, Partopour and Dixon (2016b) applied conventional reaction engineering approaches, i.e. rate-determining steps and quasi-equilibrium, to reduce a microkinetics model to a model with general rate expressions. Although the complexity was significantly reduced, deviations between the reduced and the original model occurred, especially where sharp species gradients

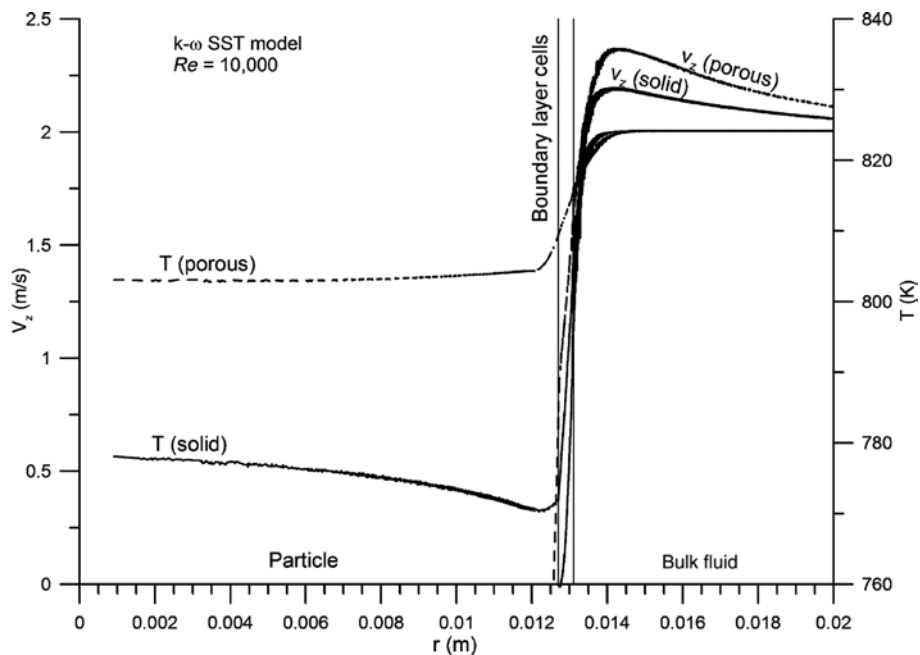


Figure 34: Comparison of temperature profiles for sphere and surrounding fluid, for porous and solid particle models for SRM. $Re = 10,000$. Reprinted with permission from Dixon et al. (2010). Copyright (2010) American Chemical Society.

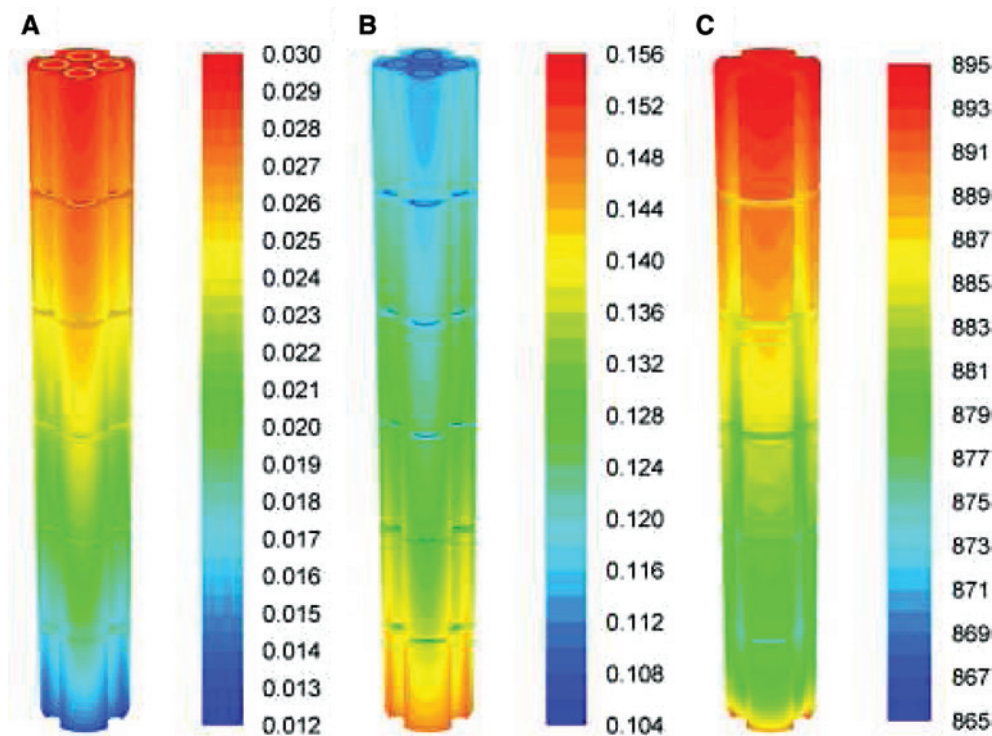


Figure 35: CFD of temperature and different mass fraction results for a stack of non-spherical pellets of SRM. (A) Hydrogen mass fraction, (B) methane mass fraction and (C) temperature (K) at the surface of the active particles in the CFD simulation of SRM. Reprinted with permission from Behnam et al. (2012). Copyright (2012) Elsevier.

appeared. In another study, Partopour and Dixon (2016a) mapped reaction rates from a microkinetics model into quadratic multivariate splines. These splines were

imported into user-defined functions and were incorporated into the CFD simulations. This study showed that although the complexity of the model was reduced, the

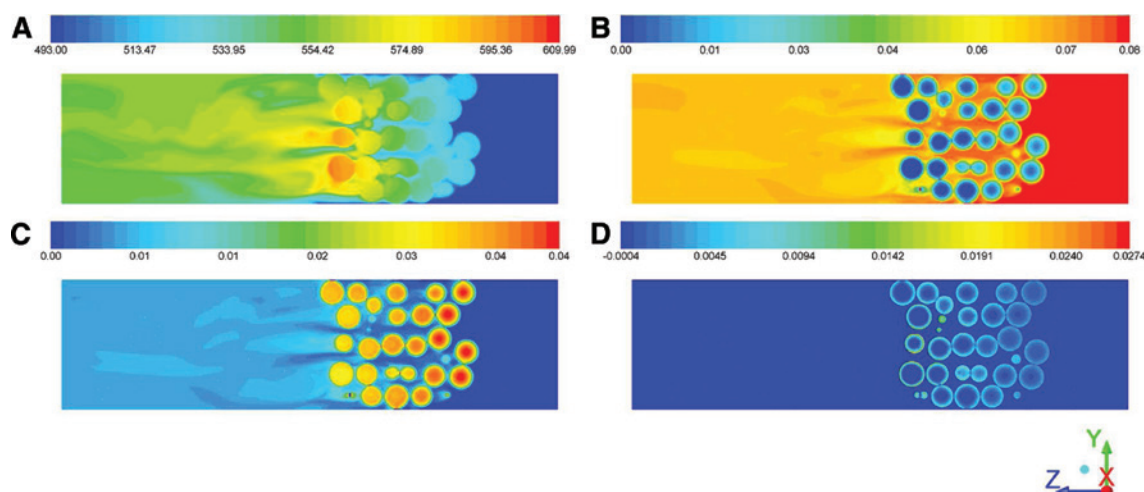


Figure 36: Solid spheres can be distinguished from the gas phase.

Contour plots of (A) temperature (K), (B) oxygen mass fraction, (C) ethyleneoxide mass fraction, and (D) reaction rate [kmol/(m³/s)] on cut through the bed. Reprinted with permission from Partopour and Dixon (2016b). Copyright (2016) American Chemical Society.

main features of the kinetic model were retained. Finally, Partopour and Dixon (2017) compared the spline method with simplified lumped rate expressions for ethylene oxidation in a fixed bed of 120 spheres ($N=5.04$). This study highlights that steep gradients inside the catalytically active spheres are the reason for significant deviations between the reduced kinetic model and the spline method with full microkinetics. This series of studies shows that not only the time saving of a method but also the accuracy or loss of information is important.

Peng et al. (2016) simulated the acetone hydrogenation in a small bed of spheres ($H/d_p=5$ high and $N=0.5, 3$) taking intraparticle heat and mass transport into account. DEM was used to generate the bed. Global shrinking of 0.5% of the spheres was conducted to avoid bad cells close to contact points. Fluent was used to generate a tetrahedral mesh, both in the gas phase and the solid phase. Prismatic meshes were used close to surfaces to account for gradient discretization. The acetone hydrogenation was described with a main reaction and a principle secondary reaction. The reaction rate constants and activation energies were developed from their own experiments. The solid particle method from Dixon et al. (2010) was used to model mass transport and chemical reactions inside the solid spheres. Knudsen and molecular diffusion was considered inside the porous spheres. Laminar flow with inlet temperature of 473 K and ambient pressure were simulated. The authors recognized large mass transport resistance inside the catalytic particle, leading to concentration peaks/sinks in the center of the spheres.

The effect of different pellet shapes towards the performance of SRM reactor ($N=1.4$) was studied by Karthik and

Buwa (2017) using CFD. Ten pellets of different shapes, i.e. cylinder, trilobe, daisy, hollow cylinder, cylcut, and seven-hole cylinder, were placed inside a tube. Global shrinking (99.5% of the actual pellet volume) was applied to overcome convergence issues in contact regions. Fluid and solid phase was meshed using tetrahedral cells accounting for 3–17 million dependent on the pellet shape. Knudsen and molecular diffusion was accounted for intraparticle mass transport. The porous media approach was used to model the catalytic pellets. This approach was criticized earlier by Dixon et al. (2010), as the no-slip condition at the solid-gas interphase is not captured properly. In order to suppress convective fluxes inside the pellets, Karthik and Buwa (2017) added a very large source term to the momentum balance equation. The Reynolds number was in the range between 2500 and 10,000 with an inlet temperature of 824 K. A LHHW kinetics from literature was used to account for the catalytic SRM. The best performing shape in terms of effective heat transfer and effectiveness factor per unit pressure drop was found to be the trilobe-shaped pellet. The authors concluded that an external shaping of a pellet leads to a lower pressure drop. However, an internal shaping of a pellet increases the accessibility to the catalyst volume within the pellet.

Maffei et al. (2016) proposed a multiregion operator-splitting approach allowing the coupling of CFD simulations with microkinetics and intraparticle transport. The computational domain is separated into fluid and porous region and solved individually. Hence, three loops are conducted, i.e. fluid region, solid region, and heterogeneous chemistry in the solid. Convergence at the boundaries is achieved via an iterative procedure, i.e. the PIMPLE algorithm. The approach is tested against experimental

data of an annular reactor. Also a fixed-bed reactor containing 50 porous spheres was simulated but without corresponding experimental data. Fuel-rich H_2 combustion on Rh described by a comprehensive microkinetics is used as a test reaction. The grid of the fixed-bed contained 585,000 fluid cells and 275,000 solid cells. Time to convergence was not reported. The intraparticle distributions of species mole fractions and adsorbed species fractions are non-uniform and affect the temperature distribution and vice versa. The underlying catalyticFOAM solver is currently bounded to transient conditions. Hence, a very small time step is necessary to achieve convergence, which leads automatically to long calculation procedures to reach steady state.

Recently, Wehinger et al. (2017b) compared different pore process models, i.e. instantaneous diffusion, effectiveness factor, and 3D reaction diffusion, coupled with

CFD and microkinetics. For the 3D approach, the so-called co-simulation model is used in the CFD software STAR-CCM+. It calculates individually fluid and solid regions allowing heat and species mass fluxes over interfaces. The different approaches were tested against experimental data obtained from a stagnation-flow reactor where CO is catalytically oxidized on a Rh washcoat. The 3D approach reproduced the experiments with high accuracy followed by the computationally cheaper effectiveness factor, see Figure 37. As internal mass transport limitations were present, the instantaneous diffusion model was not capable to predict the experiments accurately. Moreover, a catalytic single sphere was investigated with the three different approaches. Locally varying transport properties highlight the use of variable pore process modeling. Applying the 3D reaction-diffusion model inside the pellets was computationally very expensive, i.e. 2–3 orders of magnitude larger than the less sophisticated models. This study shows that for larger fixed-bed reactors, where internal mass-transport limitations are present, the calculation time of both microkinetics and pore process models has to be reduced significantly.

4 Conclusion

CFD simulations, where the actual pellet shapes are geometrically resolved, have become the most accepted detailed modeling approach for fixed-bed reactors. In the last decade, this method was further developed, and many of the current issues are investigated by different research groups. In the field of synthetic bed generation, the use of DEM simulations has turned out to be the most flexible in terms of pellet geometry and in terms of accuracy. Well-established meshing strategies include prism cells close to solid surfaces, polyhedral cells in core regions, as well as local contact treatment. The latter is still under investigation. Especially for non-spherical particles and high Reynolds numbers ($Re_p > 1000$), the local caps and local bridges method need to be evaluated in detail. A large portion of newly arisen issues is connected with coupling detailed CFD with surface reactions, which is essential to model catalytic fixed-bed reactors. One of the most fundamental questions is the description of surface reactions. Special attention is paid to efficiency and accuracy of the description, i.e. detailed reaction mechanisms (microkinetics) versus LHHW kinetics. Due to the fact that the inclusion of more detailed reaction description increases the computational time dramatically, many different strategies for a reduction of calculation time have been presented lastly. Other aspects of challenges cover the

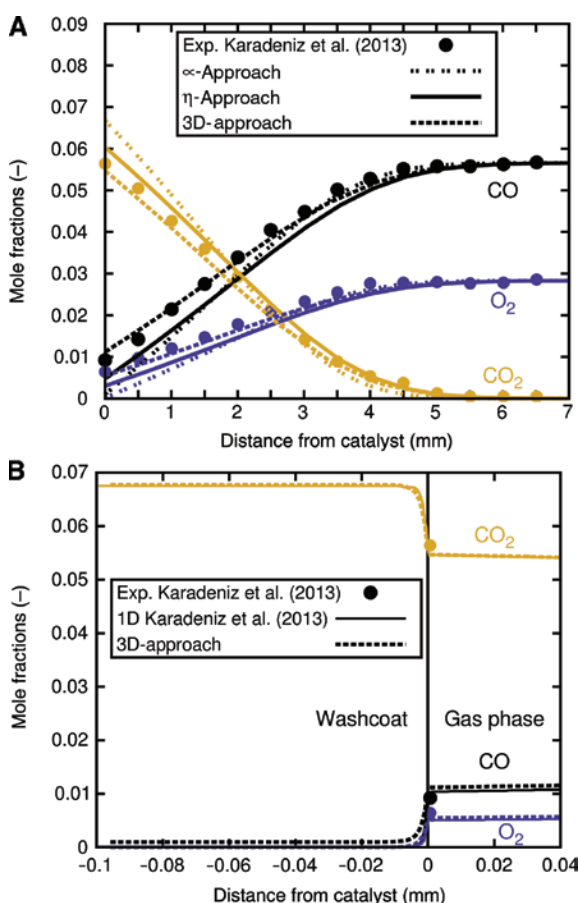


Figure 37: Mole fractions of the CO oxidation in stagnation-flow reactor. (A) Comparison between experiments from Karadeniz et al. (2013) and CFD simulations with different pore models at 873 K. (B) Mole fractions inside the washcoat at 873 K. Reprinted with permission from Wehinger et al. (2017b). Copyright (2017) Elsevier.

description of radiation between surfaces and the radiation by participating media, as many applications operate at thermal conditions up to 1000 K. Turbulence modeling in complex bed geometries is another topic, although it is not predominantly covered by chemical engineers.

This review identified five future research fields for particle-resolved CFD simulations:

1. **Bed generation by using DEM simulations.** This topic involves questions such as how can a representative bed be generated, as during operation the bed morphology changes? How do material properties, such as surface roughness, influence the bed morphology? How can a bed consisting of complex non-spherical pellets efficiently be modeled? How can artifacts resulting from composite DEM-particles, such as overlaps, be minimized?
2. **Accurate and computationally “cheap” description of surface reactions.** Research on this topic should tackle questions such as what level of detail is the most efficient for describing surface reactions by coupling them with CFD? How does an accurate and detailed kinetics look like? How can phenomena not yet being addressed, like coking, adsorbate-adsorbate interaction, and surface dynamics, be incorporated into a CFD simulation? How can the calculation time of the kinetics be reduced without losing information?
3. **Intra-particle transport.** This field of research raises questions such as how can pore processes be modeled efficiently? How to include multi-scale phenomena in the pellet into the CFD simulation of the surrounding flow field? Is a pore-resolved 3D model of the pellet of interest?
4. **Transient operations.** Many new applications will involve transient operations of fixed-bed reactors. Consequently, the question is can particle-resolved CFD simulations be applied for transient situations? If yes, when and how? What transport phenomena are dominant for a transient description? These questions lead to the last field of research, which covers the following.
5. **Model reduction.** The computational consumption of particle-resolved CFD is large and getting larger with a more detailed level of description. However, not every information of these detailed simulations is of interest. Especially for transient situations, the full CFD model is prohibitively expensive. As a consequence, we should think about reducing these detailed models or using data to feed reduced models.

Future applications of particle-resolved CFD simulations could be, for example, the model-based development

of pellet shapes or entire reactors, as well as the direct coupling of CFD and additive manufacturing. First tentative steps in this direction have already been done by Baker (2011). He used 3D printing to manufacture a computationally generated bed. Experimental pressure drop measurements and the results of CFD simulations were in good agreement, which is an indicator for the quality of the 3D printed object. The combination of CAD, CFD, optimization algorithms, and additive manufacturing into a virtual development machine could be the future of chemical engineering design.

Nomenclature

Latin letters

A	pre-exponential factor	$[1/\text{s or m}^2/\text{mol s}]$
A	area	$[\text{m}^2]$
A_w	wall area	$[\text{m}^2]$
b_k	parameter	$[-]$
c	lattice speed	$[\text{m/s}]$
c_i	molar concentration	$[\text{mol}/\text{m}^3]$
C_μ	parameter	$[-]$
d_p	particle diameter	$[\text{m}]$
d_{pore}	pore diameter	$[\text{m}]$
D	tube diameter	$[\text{m}]$
\mathbf{D}	deformation tensor	$[-]$
$D_{M,i}$	effective diffusion coefficient	$[\text{m}^2/\text{s}]$
D_{ij}	binary diffusion coefficient	$[\text{m}^2/\text{s}]$
D_{Knud}	Knudsen diffusion coefficient	$[\text{m}^2/\text{s}]$
\mathbf{e}_i	microscopic velocity	$[\text{m/s}]$
E_a	activation energy	$[\text{J/kg}]$
f	face flux	$[\text{kg}/\text{m}^2 \text{ s}]$
f_i^{eq}	local equilibrium distribution	$[-]$
f_i	probability	$[-]$
f_f	friction coefficient	$[-]$
\mathbf{F}	force	$[\text{N}]$
F_i	number of gas molecules striking the surface	$[\text{mol}/\text{m}^2 \text{ s}]$
$F_{\text{cat/geom}}$	ratio of catalytic active area to geometric area	$[-]$
\mathbf{g}	gravity	$[\text{m}/\text{s}^2]$
G	filter function	
G	Gibb's free energy	$[\text{J}/\text{mol}]$
h	specific enthalpy	$[\text{J}/\text{kg K}]$
h	Planck's constant $h=6.62607004 \cdot 10^{-34}$	$[\text{J/s}]$
h_p	particle height	$[\text{m}]$
h_w	wall heat transfer coefficient	$[\text{W}/\text{m}^2 \text{ K}]$
H	enthalpy	$[\text{J}/\text{kg}]$
j_i	diffusion mass flux	$[\text{kg}/\text{m}^2 \text{ s}]$
k	thermal conductivity	$[\text{W}/\text{m K}]$
k	turbulent kinetic energy	$[\text{J}/\text{kg}]$
k_B	Boltzmann constant	$[\text{J}/\text{K}]$
k_i	reaction rate constant	$[1/\text{s or m}^2/\text{mol s}]$
k_{eff}	effective rate coefficient	$[1/\text{s or m}^2/\text{mol s}]$
k_r	effective radial thermal conductivity	$[\text{W}/\text{m K}]$

K_i	equilibrium constant	[–]
$L()$	differential operator	
m	mass	[kg]
M_i	molecular weight	[kg/mol]
\mathbf{n}	normal vector	[m]
\bar{n}_c	mean number of contacts per particle	[–]
N	tube-to-particle diameter ratio	[–]
N_g	number of gas species	[–]
N_A	Avogadro constant $N_A = 6.02214 \cdot 10^{23}$	[1/mol]
p	pressure	[Pa]
\dot{q}	diffusive heat flux	[J/m ² s]
q_ϕ	volumetric scalar source	Varying
r	residual value	Varying
r	radial coordinate	[–]
R_i^{hom}	net production rate	[kg/m ² s]
R_i^{het}	heterogeneous reaction term	[kg/m ² s]
R	gas constant	[J/kg K]
s	variance	varying
\dot{s}_i^{ads}	adsorption rate	[kg/m ² s]
\dot{s}	molar net production rate	[mol/m ² s]
S	face area	[m ²]
S_i	sticking coefficient	[–]
S_i^0	initial sticking coefficient	[–]
S	entropy	[J/mol]
t	time	[s]
t_w	washcoat thickness	[–]
\mathbf{T}	stress tensor	[Pa]
T	temperature	[K]
\mathbf{v}	velocity	[m/s]
v_{in}	superficial velocity	[m/s]
$\bar{\mathbf{v}}$	mean velocity	[m/s]
\mathbf{v}'	fluctuating velocity	[m/s]
V	volume	[m ³]
\mathbf{x}	position vector	[m]
X_i	mole fraction	[–]
Y_i	mass fraction	[–]
z	axial coordinate	[m]

Greek letters

Γ	surface site density	[mol/m ²]
Γ	diffusive transport coefficient	Varying
γ	catalyst density	[1/m]
δ	Kronecker delta	[–]
ϵ	parameter for modified activation energy	[kJ/mol]
ϵ	void fraction	[–]
ϵ	turbulent dissipation rate	[J/kg s]
ϵ_w	washcoat porosity	[–]
η	parameter for modified pre-exponential factor	[–]
η_L	effectiveness factor	[–]
Θ	surface coverage	[–]
θ_k	shape function	
μ	dynamic viscosity	[Pa s]
μ	parameter for modified surface rate expression	[–]
ν	stoichiometric coefficient	[–]
ρ	density	[kg/m ³]

σ_i	coordinate number	[–]
$\boldsymbol{\tau}$	subgrid-scale Reynolds stress tensor	[Pa]
$\boldsymbol{\tau}'$	Reynolds stress tensor	[Pa]
$\boldsymbol{\tau}$	viscous stress tensor	[Pa]
τ_w	tortuosity	[–]
ϕ	scalar quantity	varying
ϕ	Thiele modulus	[–]
Ψ	test function	[–]
Ω	collision term	
Ω	boundaries	
ω	specific dissipation rate	[1/s]

Subscripts

ads	adsorption
b	back
b	body
b	buoyancy
c	contact
c	cell
cat	catalyst
CL	catalytically active layer
eff	effective
eq	equilibrium
eq	equivalent
exp	experimental
f	fluid
f	forward
g	gas
g	gravity
in	inlet
k	kinetic energy
Knud	Knudsen
L	local
M	mixture
n	normal
out	outlet
p	particle
r	radial
ref	reference
rot	rotational
rxn	reaction
s	surface
t	turbulent
t	tangential
trans	translational
vib	vibrational
w	wall
w	window

Superscripts

#	at the transition state
0	at standard conditions
ads	adsorption
gas	gas phase
het	heterogeneous

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